

Response of forced Euler-Bernoulli beams using differential transform method

Seval Catal*

*Department of Civil Engineering, Faculty of Engineering (Applied Mathematics),
Dokuz Eylül University, Izmir, Turkey*

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Abstract. In this paper, forced vibration differential equations of motion of Euler-Bernoulli beams with different boundary conditions and dynamic loads are solved using differential transform method (DTM), analytical solutions. Then, the modal deflections of these beams are obtained. The calculated modal deflections using DTM are represented in tables and depicted in graphs and compared with the results of the analytical solutions where a very good agreement is observed.

Keywords: differential transform method; forced vibration; partial differential equations; natural frequencies

1. Introduction

Engineering problems are usually described by ordinary and partial differential equations. Mostly these problems cannot be solved or are difficult to solve analytically. Alternatively, the numerical methods can provide approximate solution rather than the analytical solutions of problems. Forced vibration equations of Euler-Bernoulli beams are fourth-order partial differential equation.

The dynamic analysis of forced beams is one of the important topics in engineering. The behavior of the beam has been investigated by many researchers in the past. Clough and Penzien have solved the forced vibration equation of the Euler-Bernoulli beam with simply supported and subjected to concentrated constant force at mid-span using modal analysis (Clough and Penzien 1993). Chopra has derived mathematical expressions for the dynamic response of simply supported beam to a step-function force (Chopra 1995). Paz has determined the modal steady – state response of the fixed beam subjected to harmonic load (Paz 1997). Paz and Dung have derived general stiffness matrix for Euler-Bernoulli beam elements using power series (Paz and Dung 1975). Kai-Yuan *et al.* have investigated the dynamic response of Euler-Bernoulli beams with arbitrary nonhomogeneity and arbitrary variable cross-section (Kai-yuan *et al.* 1992). Fan *et al.* have proposed a method of analysis for the forced vibration of a beam with viscoelastic boundary supports (Fan *et al.* 1998). Hilal has obtained the dynamic response of prismatic damped Euler-Bernoulli beams subjected to distributed and concentrated dynamic load using Green Functions (Hilal 2003). Oniszczuk has

*Corresponding author, Ph.D., E-mail: seval.catal@deu.edu.tr

determined the steady-state response of the simply supported double-beam system using modal analysis (Oniszczuk 2003). Celebi *et al.* have obtained displacements of non-uniform rods subjected to dynamic axial load using Laplace transformation (Celebi *et al.* 2011).

Catal has calculated natural frequencies of beam on elastic soil using DTM (Catal 2006). Catal and Catal have calculated the critical buckling loads of partially embedded pile in elastic soil using DTM (Catal and Catal 2006). Catal has investigated free vibration of beam on elastic soil using DTM as well (Catal 2008). Yesilce and Catal have calculated the natural frequencies of axially loaded Reddy-Bickford beam on elastic soil using DTM (Yesilce 2009). DTM has been used to find the non-dimensional natural frequencies of the tapered cantilever Euler-Bernoulli and Timoshenko beams by Ozgumus and Kaya (2006, 2010). Yesilce has calculated natural frequency of semi-rigid connected Reddy-Bickford beam and a moving beam using DTM (Yesilce 2011, 2010). DTM has been used to find natural frequencies of elastically supported Timoshenko columns with a tip mass by Demirdag and Yesilce (2011). Yesilce and Catal have calculated natural frequencies of semi-rigid connected Reddy-Bickford beams resting on elastic soil using DTM (Yesilce and Catal 2011). Chen and Ho have proposed a method to solve eigen - value problems for the free and transverse vibration problems of rotating twisted Timoshenko beam under axial loading by using DTM (Chen and Ho 1996, 1999). DTM has been applied to solve a second-order non-linear differential equation that describes the under damped and over damped motion of a system subjected to external excitations by Jang and Chen (1997). DTM has applied to eigen-value problems and Sturm-Liouville eigen-value problem by Hassan (2002a, b). The DTM was first proposed by Zhou in 1986 and was applied to solve electric circuit analysis (Zhou 1986). In this study, a new transformation called differential transform has introduced to solve the forced vibration equations of Euler-Bernoulli beams.

2. Formulation of problem

The forced vibration equation of a uniform elastic undamped Euler-Bernoulli beam acted by lateral forces $P(x, t)$ is described by the partial differential equation as (Paz 1997)

$$EIy''(x, t) + m\ddot{y}(x, t) = P(x, t) \quad (1)$$

where EI is the flexural rigidity; t is time variable; m is the mass per unit length of the beam; x is position, $y(x, t)$ is lateral displacement function of the beam; $y''(x, t) = \partial^4 y(x, t)/\partial x^4$; $\ddot{y}(x, t) = \partial^2 y(x, t)/\partial t^2$ the lateral displacement function of the beam at position x and time t is written using separable of variables method as

$$y(x, t) = \sum_{n=1}^{\infty} Y_n(x) T_n(t) \quad (2)$$

where $Y_n(x)$ and $T_n(t)$ are the n -th normal mode of the beam, n -th generalized deflection of the beam, respectively. Substituting Eq. (2) in Eq. (1) gives

$$EIY_n''(x)T_n(t) + mY_n(x)\ddot{T}_n(t) = P(x, t) \quad (3)$$

Multiplying each term of Eq. (3) by the j -th normal mode and integrating between 0 and length of beam L yields

$$\int_0^L EIY_n^{vv}(x)Y_j(x)T_n(t)dx + \int_0^L mY_n(x)Y_j(x)\ddot{T}_n(t)dx = \int_0^L Y_j(x)P(x,t)dx \quad (4)$$

According to the orthogonality properties of normal modes, Eq. (4) is written as (Chopra 1995)

$$M_n\ddot{T}_n(t) + K_nT_n(t) = P_n \quad (5)$$

where, $\ddot{T}_n(t) = \frac{\partial^2 T_n(t)}{\partial t^2}$, $M_n = \int_0^L mY_n^2(x)dx$, $K_n = \int_0^L EIY_n^{vv}(x)Y_n(x)dx$, and $P_n = \int_0^L Y_n(x)P(x,t)dx$ are the generalized mass stiffness and force, respectively. The differential equation of free vibration depends on the n -th normal mode of the beam is given as (Clough and Penzien 1993, Chopra 1995, Paz 1997)

$$Y_n^{vv}(x) - \lambda_n^4 Y_n(x) = 0 \quad (6)$$

where $\lambda_n^4 = \frac{m\omega_n^2}{EI}$; ω_n is the natural frequency of the n -th normal mode of the beam.

3. Differential transformation

The differential transformation technique, which was first proposed by Zhou in 1986, is one of the numerical methods for ordinary and partial differential equations that use the form of polynomials as the approximation to the exact solutions that are sufficiently differentiable. $f(x, t)$ function that will be solved and the calculation of following derivatives necessary in the solution become more difficult when the order increases. This is in contrast with the traditional high-order Taylor series method. Instead, the differential transform technique provides an iterative procedure to obtain higher-order series. Therefore, it can be applied to the case of high order (Çatal 2008).

Basic definitions and operations of differential transformation are introduced and also differential transformation of the function $y(x)$ is defined as follows

$$\bar{y}(k) = \frac{1}{k!} \left[\frac{d^k y(x)}{dx^k} \right]_{x=x_0} \quad (7)$$

In Eq. (7), $y(x)$ is the original function and $\bar{y}(k)$ is transformed function which is called the T -function (it is also called the spectrum of the $y(x)$ at $x = x_0$, in the K domain). The differential inverse transformation of $\bar{y}(k)$ is defined as

$$y(x) = \sum_{k=0}^{\infty} (x-x_0)^k \bar{y}(k) \quad (8)$$

from Eq. (7) and Eq. (8), we get

$$y(k) = \sum_{k=0}^{\infty} \frac{(x-x_0)^k}{k!} \left[\frac{d^k y(x)}{dx^k} \right]_{x=x_0} \quad (9)$$

Eq. (9) implies that the concept of the differential transformation is derived from Taylor's series expansion, but the method does not evaluate the derivatives symbolically. However, relative

derivative are calculated by iterative procedure that are described by the transformed equations of the original functions.

From the definitions of Eq. (7) and Eq. (8), it is easily proven that the transformed functions comply with the basic mathematical operations shown in below. In real applications, the function $y(x)$ in Eq. (8) is expressed by a finite series and can be written as

$$y(x) = \sum_{k=0}^N (x-x_0)^k \bar{y}(k) \quad (10)$$

Eq. (10) implies that $\sum_{k=N+1}^{\infty} (x-x_0)^k \bar{y}(k)$ is negligibly small and N is decided by the converge of the displacements, where N is series size.

3.1 Some basic mathematical operations of the differential transformation

The fundamental mathematical operations performed by differential transformation are listed, where the transformed functions $\bar{y}(k)$ are related to the known original function $y(x)$, as follows

Original function $y(x)$	Transformed function $\bar{y}(k)$	
$ay(x)$	$a\bar{y}(k)$	(11.1)
$y_1(x) \pm y_2(x)$	$\bar{y}_1(k) \pm \bar{y}_2(k)$	(11.2)
$dy(x)/dx$	$(k+1)\bar{y}(k+1)$	(11.3)
$d^2y(x)/dx^2$	$(k+1)(k+2)\bar{y}(k+2)$	(11.4)
$d^3y(x)/dx^3$	$(k+1)(k+2)(k+3)\bar{y}(k+3)$	(11.5)
$d^4y(x)/dx^4$	$(k+1)(k+2)(k+3)(k+4)\bar{y}(k+4)$	(11.6)
$a\sin(\omega t)$	$a\frac{\omega^k}{k!}\sin\left(\frac{\pi}{2}k\right)$	(11.7)

3.2 Using differential transformation to solve motion equations

The n -th normal mode and generalized deflection of the both ends simply supported and one end fixed and other end simply supported beams are calculated from Eq. (6) and Eq. (5) by using DTM.

3.2.1 Both ends simply supported Euler-Bernoulli beam

The boundary conditions of the both ends simply supported beam subject to a step-function force P_0 at distance ξ form the left end and time t

$$Y_n(x=0) = 0 \quad (12.1)$$

$$Y_n(x=L) = 0 \quad (12.2)$$

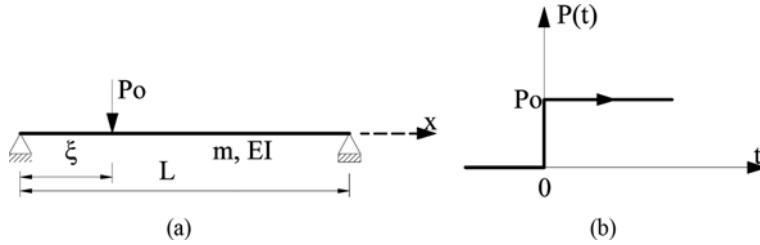


Fig. 1 (a) Both ends simply supported beam, b) the step-function force

$$\left. \frac{d^2 Y_n(x)}{dx^2} \right|_{x=0} = 0 \quad (12.3)$$

$$\left. \frac{d^2 Y_n(x)}{dx^2} \right|_{x=L} = 0 \quad (12.4)$$

$$T_n(t=0) = 0 \quad (12.5)$$

$$\left. \frac{dT_n(t)}{dt} \right|_{t=0} = 0 \quad (12.6)$$

Substituting $P(x, t) = P_0 \delta(x - \xi)$ in the generalized force gives (Chopra 1995)

$$P_n = P_0 Y_n(\xi) \quad (13)$$

where $\delta(x - \xi)$ is the Dirac delta function centered at ξ . Substituting Eq. (13) in Eq. (5) gives

$$M_n \ddot{T}_n(t) + K_n T_n(t) = P_0 Y_n(\xi) \quad (14)$$

Applying the differential transform method to Eqs. (6)-(14) and using the theorems introduced Eqs. (11.1)-(11.6), the following expression are obtained.

$$\bar{Y}_n(k+4) = \frac{\lambda^4 \bar{Y}_n(k)}{(k+1)(k+2)(k+3)(k+4)}, \quad (k=0, 1, 2, 3, \dots) \quad (15)$$

$$\bar{T}_n(k+2) = -\left[\frac{P_0 Y_n(\xi) \beta(k) - \lambda^4 \bar{T}_n(k)}{(k+1)(k+2)} \right], \quad (k=0, 1, 2, 3, \dots) \quad (16)$$

$$\text{where } \beta(k) = \begin{cases} 1, & \text{if } k=0 \\ 0, & \text{if } k \neq 0 \end{cases}$$

Boundary conditions (12.1) and (12.2) are written using Eq. (10) for near the $x_0 = 0$ point as in following respectively

$$\text{for } x = 0; \quad \bar{Y}_n(0) = 0 \quad (17)$$

$$\text{for } x = L; \quad \bar{Y}_n(1) \sum_{k=0}^N \frac{L^{(4k+1)}}{(4k+1)!} (\lambda_n^4)^k + 3! \bar{Y}_n(3) \sum_{k=0}^N \frac{L^{(4k+3)}}{(4k+3)!} (\lambda_n^4)^k = 0 \quad (18)$$

Boundary conditions (12.3) and (12.4) are written using the function obtained by derivating Eq. (10) twice as in following

$$\text{for } x = 0; \quad \bar{Y}_n(2) = 0 \quad (19)$$

$$\text{for } x = L; \quad (\lambda_n^4) \bar{Y}_n(1) \sum_{k=0}^N \frac{L^{(4k+3)}}{(4k+3)!} (\lambda_n^4)^k + 3! \bar{Y}_n(3) \sum_{k=0}^N \frac{L^{(4k+1)}}{(4k+1)!} (\lambda_n^4)^k = 0 \quad (20)$$

Boundary condition (12.5) is written using Eq. (10) for near the $t_0 = 0$ time as in following

$$\text{for } t = 0; \quad \bar{T}_n(0) = 0 \quad (21)$$

Boundary condition (12.6) is written using the function obtained by derivating Eq. (10) as in following

$$\text{for } t = 0; \quad \bar{T}_n(1) = 0 \quad (22)$$

Thus,

$$\bar{T}_n(t) = P_0 Y_n(\xi) \sum_{k=0}^N \frac{t^{(2k+2)}}{(2k+2)!} (\lambda_n^4)^k (-1)^{(k+1)} \quad (23)$$

The matrix expression is obtained using Eqs. (18) and (20)

$$\begin{bmatrix} A_n & B_n \\ \lambda^4 B_n & A_n \end{bmatrix} \begin{Bmatrix} \bar{Y}_n(1) \\ 3! \bar{Y}_n(3) \end{Bmatrix} = \begin{Bmatrix} 0 \\ 0 \end{Bmatrix} \quad (24)$$

The frequency equation of the beam is obtained using determinant of the coefficients matrix of Eq. (24) as in the following

$$f_n = A_n^2 - \lambda^4 B_n^2 = 0 \quad (25)$$

Solving Eq. (25) we get $\omega = \omega_i^{(N)}$, $i = 1, 2, 3, \dots$ where $A_n = \sum_{k=0}^N \frac{L^{(4k+1)}}{(4k+1)!} (\lambda_n^4)^k$, $B_n = \sum_{k=0}^N \frac{L^{(4k+3)}}{(4k+3)!} (\lambda_n^4)^k$,

$\omega_i^{(N)}$, is the N -th estimated ω natural frequency corresponding to N , and N is indicated by

$$\left| \frac{\omega_i^{(N)} - \omega_i^{(N-1)}}{\omega_i^{(N)}} \right| \leq \varepsilon \quad (26)$$

where $\omega_i^{(N-1)}$ is the i -th estimated circular frequency corresponding to $N-1$ and ε is a positive and small value (i.e., $\varepsilon = 0,0001$).

Substituting Eqs. (15), (17) and (19) in Eq. (10) gives

$$Y_n(x) = \sum_{k=0}^N (\lambda_n^4)^k \left[\frac{x^{(4k+1)}}{(4k+1)!} \bar{Y}_n(1) + 3! \sum_{k=0}^N \frac{x^{(4k+3)}}{(4k+3)!} \bar{Y}_n(3) \right] \quad (27)$$

$\bar{Y}_p(3)$ is obtained using Eq. (24) as in the following

$$\bar{Y}_p(3) = -\frac{A_n + \lambda^4 B_n}{3!(A_n + B_n)} \bar{Y}_p(1) \quad (28)$$

Substituting Eq. (28) in Eq. (27) gives

$$Y_n(x) = \bar{Y}_n(1) \left\{ \sum_{k=0}^N (\lambda^4)^k \left[\frac{x^{(4k+1)}}{(4k+1)!} - \frac{A_n + \lambda^4 B_n}{A_n + B_n} \frac{x^{(4k+3)}}{(4k+3)!} \right] \right\} \quad (29)$$

where, value of $\bar{Y}_n(1)$ is arbitrary.

The natural frequencies, values of $Y_n(\xi)$ and $y(x, t)$ are calculated using Eq. (25), Eq. (29) and Eq. (2), respectively.

3.2.2 One end fixed and other end simply supported Euler-Bernoulli beam

The boundary conditions of the one end fixed and other end simply supported beam subjected to a harmonic load $P(x, t) = P_0 \sin(\varpi t)$ uniformly distributed along span in (Fig. 2a-2b) are given below.

$$Y_n(x = 0) = 0 \quad (30.1)$$

$$\left. \frac{dY_n}{dx} \right|_{x=0} = 0 \quad (30.2)$$

$$Y_n(x = L) = 0 \quad (30.3)$$

$$\left. \frac{d^2 Y_n}{dx^2} \right|_{x=L} = 0 \quad (30.4)$$

Dividing each term of Eq. (5) by M_n yields.

$$\ddot{T}_n(t) + \omega_n^2 T_n(t) = \frac{P_0}{m} \sin(\varpi t) C_n \quad (31)$$

where ϖ is circular frequency of the harmonic load

$$C_n = \frac{\int_0^L Y_n(x) dx}{\int_0^L Y_n^2(x) dx}$$

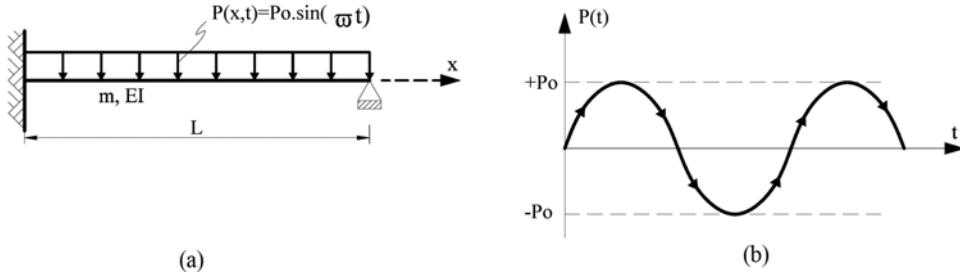


Fig. 2 (a) One end fixed and other end simply supported beam, (b) the harmonic load

The values of C_n (C_1, C_2, \dots, C_5) for the first five modes of the beam were calculated by Paz as below (Paz 1997)

$$C_1 = 0.8600; C_2 = 0.826; C_3 = 0.3345; C_4 = 0.0435; C_5 = 0.2076$$

The differential transform method is applied to Eq. (31) by using theorems introduced Eq. (11.1)-(11.7), the following expression is obtained

$$\bar{T}_n(k+2) = \frac{1}{(k+1)(k+2)} \left[\frac{C_n P_0(\omega_n)^k}{m(k!)} \sin\left(\frac{\pi k}{2}\right) - (\omega)^2 \bar{T}_n(k) \right], \quad (k=0, 1, 2, 3, \dots) \quad (32)$$

Boundary conditions (30.1) and (30.3) are written using Eq. (10) for near the $x_0 = 0$ point as following respectively

$$\text{For } x = 0; \quad \bar{Y}_n(0) = 0 \quad (33)$$

$$\text{For } x = L; \quad 2! \bar{Y}_n(2) \sum_{k=0}^N (\lambda^4)^k \frac{L^{(4k+2)}}{(4k+2)!} + 3! \bar{Y}_n(3) \sum_{k=0}^N \frac{L^{(4k+3)}}{(4k+3)!} (\lambda^4)^k = 0 \quad (34)$$

Boundary conditions (30.2) and (30.4) are written using the function obtained by deriving Eq. (10) twice as in the following

$$\text{For } x = 0; \quad \bar{Y}_n(1) = 0 \quad (35)$$

$$\text{For } x = L; \quad 2! \bar{Y}_n(2) \sum_{k=0}^N (\lambda^4)^k \frac{L^{4k}}{(4k)!} + 3! \bar{Y}_n(3) \sum_{k=0}^N \frac{L^{(4k+1)}}{(4k+1)!} (\lambda^4)^k = 0 \quad (36)$$

According to boundary conditions (12.5) and (12.6), $\bar{T}_n(0) = 0$ and $\bar{T}_n(1) = 0$ are calculated. Thus,

$$T_n(t) = \frac{C_n P_0}{m} \left\{ \sum_{k=1}^N (-1)^k \frac{t^{(2k+1)}}{(2k+1)!} \left[\sum_{h=1}^N (\varpi)^{(2k-2h+1)} \omega_n^{(2n-2)} \right] \right\} \quad (37)$$

The matrix expression is obtained using Eqs. (34) and (36)

$$\begin{bmatrix} D_n & F_n \\ G_n & H_n \end{bmatrix} \begin{bmatrix} 2! \bar{Y}_n(2) \\ 3! \bar{Y}_n(3) \end{bmatrix} = \begin{bmatrix} 0 \\ 0 \end{bmatrix} \quad (38)$$

The frequency equation of the beam is obtained using determinant of the coefficients matrix of Eq. (38) as in the following

$$F_n = D_n \cdot H_n - G_n \cdot F_n \quad (39)$$

Solving Eq. (39) we get $\omega = \omega_i^{(N)}$, $i = 1, 2, 3, \dots$ where $D_n = \sum_{k=0}^N \frac{L^{(4k+2)}}{(4k+2)!} (-\lambda^4)^k (-1)^k$, $F_n = \sum_{k=0}^N \frac{L^{(4k+3)}}{(4k+3)!} (-\lambda^4)^k (-1)^k$, $G_n = \sum_{k=0}^N \frac{L^{(4k)}}{(4k)!} (-\lambda^4)^k (-1)^k$, $H_n = \sum_{k=0}^N \frac{L^{(4k+1)}}{(4k+1)!} (-\lambda^4)^k (-1)^k$, $\omega_i^{(N)}$ is the N -th

estimated ω natural frequency corresponding to N substituting Eqs. (15), (33) and (35) in Eq. (10) gives

$$Y_n(x) = \sum_{k=0}^N (\lambda^4)^k \left[2! \frac{x^{(4k+2)}}{(4k+2)!} \bar{Y}_n(2) + 3! \frac{x^{(4k+3)}}{(4k+3)!} \bar{Y}_n(3) \right] \quad (40)$$

$\bar{Y}_p(3)$ is obtained using Eq. (38) as in the following

$$\bar{Y}_p(3) = -\frac{D_n + G_n}{3(F_n + H_n)} \bar{Y}_p(2) \quad (41)$$

Substituting Eq. (41) in Eq. (40) gives

$$Y_n(x) = \bar{Y}_n(2) \left\{ \sum_{k=0}^N (\lambda^4)^k \left[2! \frac{x^{(4k+2)}}{(4k+2)!} - 2! \left(\frac{D_n + G_n}{F_n + H_n} \right) \frac{x^{(4k+3)}}{(4k+3)!} \right] \right\} \quad (42)$$

where, value of $\bar{Y}_p(2)$ is arbitrary.

The natural frequencies and $y(x, t)$ of the beam are calculated Eq. (39) and Eq. (2), respectively.

4. Analytical solution of equations of motion

The n -th normal modes and generalized deflections of both ends simply supported beam subject to a step function force P_0 were calculated by Chopra as below (Chopra 1995)

$$Y_n(x) = \sin\left(\frac{n\pi x}{L}\right) \quad (43)$$

$$T_n(x) = \frac{2P_0 L^3}{\pi^4 EI} \frac{Y_n(\xi)}{n^4} [1 - \cos(\omega_n t)] \quad (44)$$

$$\text{where } \omega_n = n^2 \pi^2 \sqrt{\frac{EI}{mL^4}}$$

Solving Eq. (6) we get

$$Y_n(x) = A_1 \sin(\lambda x) + A_2 \cos(\lambda x) + A_3 \sinh(\lambda x) + A_4 \cosh(\lambda x) \quad (45)$$

where A_1, \dots, A_4 are constants of integration.

The constants of integration of one end fixed and other end simply supported beam are calculated using Eqs. (30.1)-(30.4) as below

$$A_3 = -A_1; \quad A_2 = -A_4; \quad A_2 = -\frac{\cos(\lambda L) - \cosh(\lambda L)}{\sin(\lambda L) - \sinh(\lambda L)} \quad (46)$$

Substituting Eq. (46) in Eq. (45) gives the n -th normal mode of one end fixed and other end simply supported beam.

$$Y_n(x) = A_1 \left\{ \cos(\lambda x) - \cosh(\lambda x) - \frac{\cos(\lambda L) - \cosh(\lambda L)}{\sin(\lambda L) - \sinh(\lambda L)} [\sin(\lambda L) - \sinh(\lambda L)] \right\} \quad (47)$$

If Eq. (31) is calculated using boundary conditions (12.5) and (12.6), the n -th generalized deflection of one end fixed and other end simply supported beam is obtained as

$$T_n(t) = \frac{P_0 C_0}{m(\omega_n^2 - \varpi^2)} \left[\sin(\varpi t) - \frac{\varpi}{\omega_n} \sin(\omega_n t) \right] \quad (48)$$

where ω_n is natural frequencies of the beam ($n = 1, 2, 3, \dots$).

The values of ω_n were calculated for the first five modes of the beam by Chopra as below (Chopra 1995)

$$\omega_1 = 15.4118\alpha \quad (49.1)$$

$$\omega_2 = 49.9648\alpha \quad (49.2)$$

$$\omega_3 = 104.2477\alpha \quad (49.3)$$

$$\omega_4 = 178.2697\alpha \quad (49.4)$$

$$\omega_5 = 272.0309\alpha \quad (49.5)$$

$$\text{where } \alpha = \sqrt{\frac{EI}{mL^4}}.$$

The lateral displacement function of both ends simply supported beam and one end fixed and other end simply supported are calculated using Eq. (43), Eq. (44), Eq. (47), Eq. (48), Eq. (2), respectively.

5. Numerical examples

For numerical examples, both ends simply supported and one end fixed and other end simply supported beams shown in Fig. 1 and Fig. 2 are considered in this study. The normal modes, the generalized deflections and the lateral displacements of the both ends simply supported beam subjected to a step-function force, $P_0 = 9.806$ kN, at distance $\xi = 2.5$ m and $\xi = 5.0$ m from the left end are calculated for the first seven modes using DTM and analytical method. The normal modes, the generalized deflections and the lateral displacements of one end fixed and other end simply supported beam subjected to a harmonic load $P(x, t) = 29.418 \sin(5t)$ kN/m uniformly distributed along the span are calculated for the first five modes using DTM and analytical method. For calculations, a computer program is established by author.

The numerical results are obtained based on a uniform beam with the following data as:

$$I = 5.73 * 10^{-6} \text{ m}^4; EI = 1179.955 \text{ kNm}^2; m = 19.612 \text{ kNs}^2/\text{m}^2; L = 10 \text{ m}$$

The lateral displacements at $x = 1$ m, 2 m, ..., 10 m points and $t = 0.1, 0.2, \dots, 1.0$ s time of the both ends simply supported and beam subjected to a step-function force ($P_0 = 9.906$ kN) at distance $\xi = 2.5$ m and $\xi = 5.0$ m from the left end are presented in (Tables 1, 2) by using DTM and analytical method, respectively.

Both analytical solution and DTM solution obtained the both ends simply supported beam at $N = 50$ are plotted in Figs. 3-6 for $\xi = 2.5$ m and $\xi = 5.0$ m, respectively.

The lateral displacements at $x = 1$ m, 2 m, ..., 10 m points of one end fixed and other end simply supported beam subjected to $P(x, t) = 29.418 \sin(5t)$ kN/m are presented in Table 3 by using DTM and analytical method.

Both the analytical solution and DTM solution obtained as $N = 50$ of one end fixed and other end simply supported beam are plotted in Figs. 7, 8, respectively.

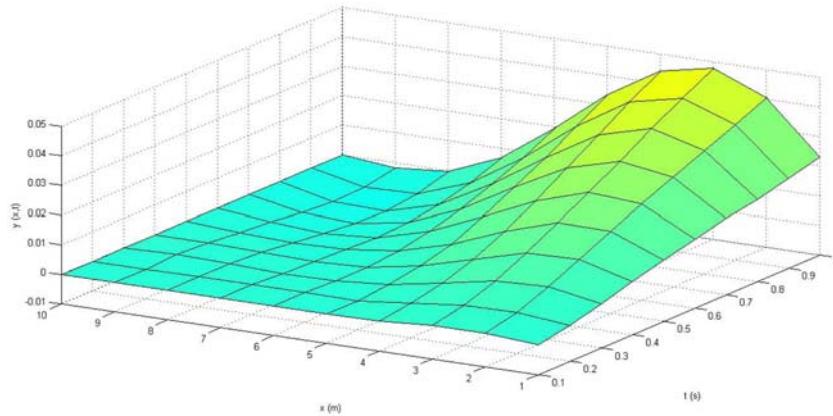


Fig. 3 Variation of the lateral displacements of both ends simply supported beam due to time and position for $\xi = 2.5$ m (DTM)

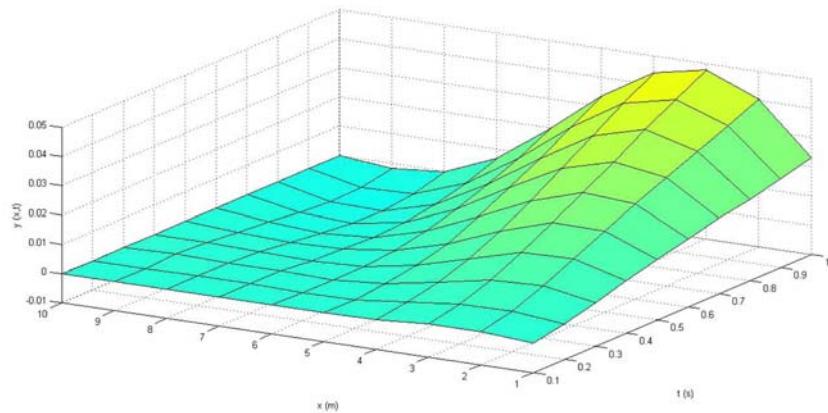


Fig. 4 Variation of the lateral displacements of both ends simply supported beam due to time and position for $\xi = 2.5$ m (Analytical Method)

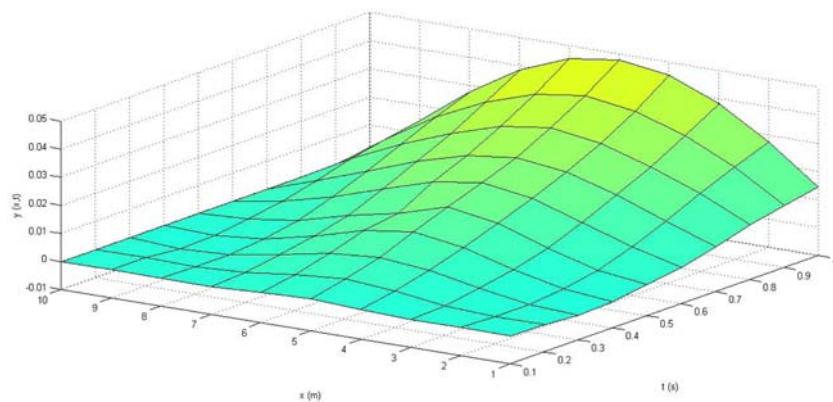


Fig. 5 Variation of the lateral displacements of both ends simply supported beam due to time and position for $\xi = 5.0$ m (DTM)

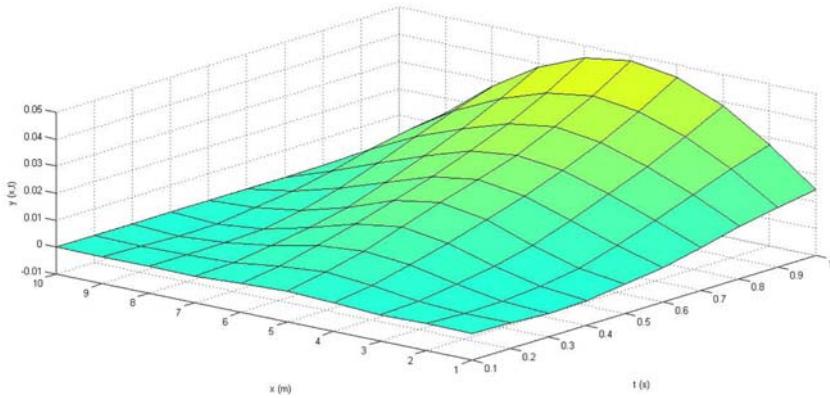


Fig. 6 Variation of the lateral displacements of both ends simply supported beam due to time and position for $\xi = 5.0$ m (Analytical Method)

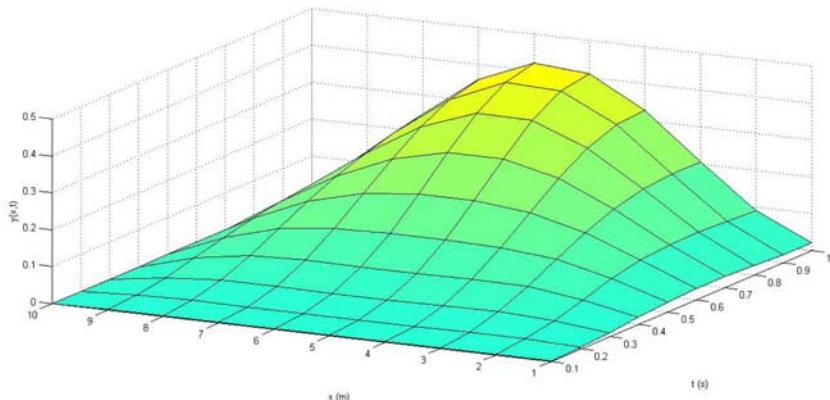


Fig. 7 Variation of the lateral displacements of one ends fixed and other end simply supported beam due to time and position (DTM)

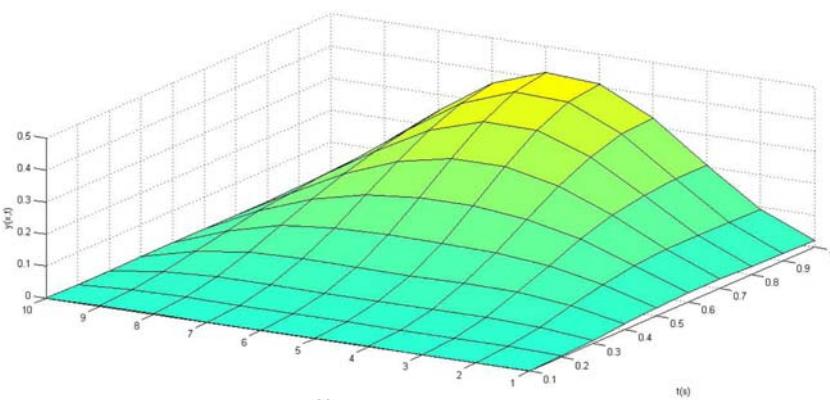


Fig. 8 Variation of the lateral displacements of one ends fixed and other end simply supported beam due to time and position (Analytical Method)

Table 1(a) The lateral displacement of the both ends simply supported beam subjected to $P_0 = 9.806$ kN at $\xi = 2.5$ m

x (m)	Method	N	$t = 0.1$ s.	$t = 0.2$ s.	$t = 0.3$ s.	$t = 0.4$ s.	$t = 0.5$ s.	$t = 0.6$ s.	$t = 0.7$ s.	$t = 0.8$ s.	$t = 0.9$ s.	$t = 1.0$ s.	
1.0	DTM	10	0.000123	0.002152	0.004347	-0.56492	-104.987	-7148.13	-246473	-5184.156	-7499046	-80942353	
		12	0.000102	0.002145	0.004958	-0.08071	-40.8075	-5928.36	-388614	-142617.7	-33683488	-56.31e+8	
		30	0.000102	0.002145	0.004989	0.007854	0.010614	0.013263	0.014110	-8.713450	-15664.57	-126504.1	
		40	0.000102	0.002145	0.004989	0.007854	0.010614	0.013263	0.015880	0.017566	0.017926	-16.09983	
		42	0.000102	0.002145	0.004989	0.007854	0.010614	0.013263	0.015880	0.017566	0.020255	-0.553427	
		45	0.000102	0.002145	0.004989	0.007854	0.010614	0.013263	0.015880	0.017566	0.020311	0.025693	
		50	0.000102	0.002145	0.004989	0.007854	0.010614	0.013263	0.015880	0.017566	0.020311	0.022914	
		51	0.000102	0.002145	0.004989	0.007854	0.010614	0.013263	0.015880	0.017566	0.020311	0.022914	
	Analytic Method		0.000103	0.002146	0.004991	0.007855	0.010614	0.013262	0.015880	0.017566	0.020310	0.022914	
	2.0	10	0.001214	0.003863	0.008841	0.684444	123.1117	8379.441	288904.5	6076162	878876.04	9485742	
		12	0.001239	0.003872	0.008124	0.116826	47.91429	6956.355	455991.2	167342.06	3952234.1	66.06e+8	
		30	0.001239	0.003872	0.008088	0.012898	0.017700	0.022588	0.028967	10.27962	18386.604	148486.2	
		40	0.001239	0.003872	0.008088	0.012898	0.017700	0.022588	0.028967	0.031455	0.038624	18.964788	
		42	0.001239	0.003872	0.008088	0.012898	0.017700	0.022588	0.026888	0.031455	0.035890	0.716940	
		45	0.001239	0.003872	0.008088	0.012898	0.017700	0.022588	0.026888	0.031455	0.035822	0.037188	
		50	0.001239	0.003872	0.008088	0.012898	0.017700	0.022588	0.026888	0.031455	0.035822	0.040442	
		51	0.001239	0.003872	0.008088	0.012898	0.017700	0.022588	0.026888	0.031455	0.035822	0.040442	
		Analytic Method	0.001239	0.003873	0.008089	0.012898	0.017699	0.022588	0.026888	0.031454	0.035821	0.040442	
		3.0	10	0.001251	0.003718	0.007121	-0.20068	-39.0147	-2656.03	-91540.3	-1924.573	-278288.2	-30027402
			12	0.001243	0.003715	0.007348	-0.02105	-15.3739	-2235.31	-146517	-5376.627	-126975.6	-21.22e+8
			30	0.001243	0.003715	0.007360	0.012342	0.018147	0.023861	0.029140	-3.261580	-5916.892	-4778.39
			40	0.001243	0.003715	0.007360	0.012342	0.018147	0.023861	0.029808	0.036357	0.041119	-6.04225
			42	0.001243	0.003715	0.007360	0.012342	0.018147	0.023861	0.029804	0.036352	0.041998	-0.169958
			45	0.001243	0.003715	0.007360	0.012342	0.018147	0.023861	0.029804	0.036352	0.042014	0.048791
			50	0.001243	0.003715	0.007360	0.012342	0.018147	0.023861	0.029804	0.036352	0.042014	0.047736
			51	0.001243	0.003715	0.007360	0.012342	0.018147	0.023861	0.029804	0.036352	0.042014	0.047736
		Analytic Method	0.001242	0.003714	0.007358	0.012340	0.018146	0.023861	0.029804	0.036352	0.042014	0.047736	

Table 1(b) The lateral displacement of the both ends simply supported beam subjected to $P_0 = 9.806$ kN at $\xi = 2.5$ m

x (m)	Method	N	$t = 0.1$ s.	$t = 0.2$ s.	$t = 0.3$ s.	$t = 0.4$ s.	$t = 0.5$ s.	$t = 0.6$ s.	$t = 0.7$ s.	$t = 0.8$ s.	$t = 0.9$ s.	$t = 1.0$ s.	
4.0	DTM	10	0.000151	0.001728	0.003539	-0.39664	-74.1017	-5054.98	-174002	-3660.090	-529475.2	-57152880	
		12	0.000134	0.001722	0.003969	-0.05707	-29.8064	-4330.91	-283902	-104190.4	-24607984	-41.13e+8	
		30	0.000134	0.001722	0.003990	0.007625	0.012542	0.018200	0.023660	-6.345713	-11441.43	-9239.902	
		40	0.000134	0.001722	0.003990	0.007625	0.012542	0.018200	0.024955	0.031450	0.036480	-11.73191	
		42	0.000134	0.001722	0.003990	0.007625	0.012542	0.018200	0.024960	0.031450	0.038182	-0.376767	
		45	0.000134	0.001722	0.003990	0.007625	0.012542	0.018200	0.024954	0.031444	0.038225	0.046224	
		50	0.000134	0.001722	0.003990	0.007625	0.012542	0.018200	0.024954	0.031444	0.038216	0.044190	
		51	0.000134	0.001722	0.003990	0.007625	0.012542	0.018200	0.024954	0.031444	0.038216	0.044190	
	Analytic Method		0.000133	0.001720	0.003989	0.007624	0.012540	0.018199	0.024955	0.031443	0.038216	0.044190	
	Analytic Method	10	-0.00017	-0.00010	0.001926	0.839119	153.3074	10434.80	359736.4	7565237	10941771	11.81e+8	
		12	-0.00015	-0.00010	0.001018	0.112031	50.40083	7319.140	479764.7	176063.75	4158156.9	69.50e+8	
		30	-0.00015	-0.00010	0.000980	0.002680	0.005440	0.010199	0.017752	10.80443	19346.570	15623.878	
		40	-0.00015	-0.00010	0.000980	0.002680	0.005440	0.010198	0.015567	0.021200	0.0303690	19.945815	
		42	-0.00015	-0.00010	0.000980	0.002680	0.005440	0.010195	0.015565	0.021199	0.0274920	0.745237	
		45	-0.00015	-0.00010	0.000980	0.002680	0.005440	0.010195	0.015565	0.021197	0.0127423	0.029995	
		50	-0.00015	-0.00010	0.000980	0.002680	0.005440	0.010195	0.015565	0.021197	0.0274190	0.033418	
		51	-0.00015	-0.00010	0.000980	0.002680	0.005440	0.010195	0.015565	0.021197	0.0274190	0.033418	
		Analytic Method	-0.00015	-0.00010	0.000980	0.002679	0.005438	0.010195	0.015564	0.021196	0.0274190	0.033418	
			10	0.000158	-0.00045	0.000339	0.771767	141.520	9634.115	332176.4	6986494.58	10105816	10.91e+8
			12	0.000032	-0.00049	-0.00061	-0.06373	-29.1780	-4237.55	-277756	-1019267	-2407141	-40.23e+8
			30	0.000032	-0.00049	-0.00059	-0.00042	0.000652	0.002781	0.004372	-6.27009	-112266.8	-90988.94
			40	0.000032	-0.00049	-0.00059	-0.00042	0.000652	0.002781	0.005645	0.009750	0.012354	-11.57682
			42	0.000032	-0.00049	-0.00059	-0.00042	0.000652	0.002781	0.005645	0.009750	0.014029	-0.394966
			45	0.000032	-0.00049	-0.00059	-0.00042	0.000652	0.002781	0.005645	0.009750	0.014070	0.021570
			50	0.000032	-0.00049	-0.00059	-0.00042	0.000652	0.002781	0.005645	0.009750	0.014070	0.019575
			51	0.000032	-0.00049	-0.00059	-0.00042	0.000652	0.002781	0.005645	0.009750	0.014070	0.019575
			Analytic Method	0.000032	-0.00048	-0.00059	-0.00042	0.000651	0.002780	0.005645	0.009749	0.014068	0.019571

Table 1(c) The lateral displacement of the both ends simply supported beam subjected to $P_0 = 9.806$ kN at $\xi = 2.5$ m

Table 1(d) The lateral displacement of the both ends simply supported beam subjected to $P_0 = 9.806$ kN at $\xi = 2.5$ m

Table 2(a) The lateral displacement of the both ends simply supported beam subjected to $P_0 = 9.806$ kN at $\xi = 5$ m.

x (m)	Method	N	$t = 0.1$ s.	$t = 0.2$ s.	$t = 0.3$ s.	$t = 0.4$ s.	$t = 0.5$ s.	$t = 0.6$ s.	$t = 0.7$ s.	$t = 0.8$ s.	$t = 0.9$ s.	$t = 1.0$ s.	
1.0	DTM	10	0.000042	-0.00031	-0.00221	-0.95477	-174.857	-11902.1	-410320	-862.9023	-12480354	-13.47e+8	
		12	0.00026	-0.00031	-0.00118	-0.12558	-57.5441	-8357.51	-547829	-201042.3	-47480843	-79.37e+8	
		30	0.000026	-0.00031	-0.00113	-0.00071	0.001030	0.002835	0.003422	-12.30349	-22091.24	-17840.45	
		40	0.000026	-0.00031	-0.00113	-0.00071	0.001030	0.002841	0.005915	0.009565	0.008851	-22.72296	
		42	0.000026	-0.00031	-0.00113	-0.00071	0.001030	0.002857	0.005917	0.009565	0.012136	-0.798381	
		45	0.000026	-0.00031	-0.00113	-0.00071	0.001030	0.002867	0.005927	0.009575	0.012228	0.018333	
		50	0.000026	-0.00031	-0.00113	-0.00071	0.001030	0.002867	0.005927	0.009575	0.012228	0.014435	
		51	0.000026	-0.00031	-0.00113	-0.00071	0.001030	0.002867	0.005927	0.009575	0.012228	0.014435	
	Analytic Method		0.000026	-0.00031	-0.00113	-0.00071	0.001030	0.002869	0.005928	0.009579	0.012230	0.014436	
	Analytic Method	10	-0.00006	-0.00041	0.000990	1.121090	205.3083	13974.48	481766.3	101315.2	14653458	15.81e+8	
		12	-0.00004	-0.00040	-0.00023	0.147479	67.54782	9809.772	643024.5	235976.7	5573143	93.16e+8	
		30	-0.00004	-0.00040	-0.00027	0.000940	0.003330	0.007407	0.015226	14.470358	25929.985	209405.15	
		40	-0.00004	-0.00040	-0.00027	0.000936	0.003332	0.007410	0.012310	0.017718	0.026927	26.715991	
		42	-0.00004	-0.00040	-0.00027	0.000917	0.003340	0.007421	0.012310	0.017729	0.023071	0.981663	
		45	-0.00004	-0.00040	-0.00027	0.000925	0.003343	0.007425	0.012310	0.017730	0.022999	0.023031	
		50	-0.00004	-0.00040	-0.00027	0.000925	0.003343	0.007425	0.012310	0.017730	0.022999	0.027637	
		51	-0.00004	-0.00040	-0.00027	0.000925	0.003343	0.007425	0.012310	0.017730	0.022999	0.027654	
		Analytic Method	-0.00004	-0.00040	-0.00027	0.000925	0.003345	0.007425	0.012317	0.017731	0.023006	0.027655	
		Analytic Method	10	-0.00014	0.000588	0.002197	-0.35278	-65.5480	-4462.19	-153833	-3235.112	-467901.8	-50497477
			12	-0.00015	0.000586	0.002585	-0.04226	-21.7278	-3156.80	-206927	-7593.813	-17934569	-29.98e+8
			30	-0.00015	0.000590	0.002601	0.004903	0.008162	0.013100	0.017720	-4.626617	-8344.424	-6738814
			40	-0.00015	0.000590	0.002601	0.004903	0.008162	0.013100	0.018660	0.024350	0.029882	-8.550131
			42	-0.00015	0.000590	0.002601	0.004903	0.008162	0.013100	0.018663	0.024354	0.031123	-0.268632
			45	-0.00015	0.000590	0.002601	0.004903	0.008162	0.013100	0.018668	0.024380	0.031179	0.039862
			50	-0.00015	0.000590	0.002601	0.004903	0.008162	0.013100	0.018668	0.024380	0.031179	0.038410
			51	-0.00015	0.000590	0.002601	0.004903	0.008162	0.013100	0.018668	0.024380	0.031179	0.038410
			Analytic Method	-0.00015	0.000591	0.002609	0.004914	0.008176	0.01311	0.018668	0.024380	0.031181	0.038411

Table 2(b) The lateral displacement of the both ends simply supported beam subjected to $P_0 = 9.806$ kN at $\xi = 5$ m

x (m)	Method	N	$t = 0.1$ s.	$t = 0.2$ s.	$t = 0.3$ s.	$t = 0.4$ s.	$t = 0.5$ s.	$t = 0.6$ s.	$t = 0.7$ s.	$t = 0.8$ s.	$t = 0.9$ s.	$t = 1.0$ s.	
4.0	DTM	10	0.000693	0.002874	0.005294	-0.66268	-123.249	-8390.17	-289249	-6082.904	-8797844	-94949147	
		12	0.000679	0.002873	0.006021	-0.08134	-42.0146	-6103.99	-400114	-146833.7	-346782.2	-57.96e+8	
		30	0.000680	0.002872	0.006053	0.009860	0.014960	0.018675	0.022007	-8.963690	-16135.48	-130307.2	
		40	0.000680	0.002872	0.006053	0.009860	0.014960	0.018675	0.023820	0.029809	0.034493	-16.56196	
		42	0.000680	0.002872	0.006053	0.009860	0.014960	0.018675	0.023830	0.029809	0.036892	-0.548182	
		45	0.000680	0.002872	0.006053	0.009860	0.014960	0.018675	0.023840	0.029829	0.036969	0.048348	
		50	0.000680	0.002872	0.006053	0.009860	0.014960	0.018675	0.023840	0.029829	0.036969	0.045500	
		51	0.000680	0.002872	0.006053	0.009860	0.014960	0.018675	0.023840	0.029829	0.036969	0.045500	
	Analytic Method		0.000682	0.002873	0.006061	0.009865	0.014969	0.018671	0.023847	0.029830	0.036968	0.045506	
			10	0.001475	0.004232	0.009276	1.409783	256.1348	17433.17	601003.2	126390.7	1828018.1	19.72e+8
5.0	DTM	12	0.001475	0.004232	0.007740	0.166605	71.10925	10325.04	676799	24837.12	58658694	98.05e+8	
		30	0.001475	0.004232	0.007686	0.012345	0.017021	0.021241	0.029016	15.239438	27283.821	220338.33	
		40	0.001475	0.004232	0.007686	0.012345	0.017021	0.021241	0.025934	0.032206	0.043501	28.12977	
		42	0.001475	0.004232	0.007686	0.012345	0.017021	0.021241	0.025934	0.032206	0.039444	1.051836	
		45	0.001475	0.004232	0.007686	0.012345	0.017021	0.021241	0.025934	0.032206	0.039346	0.043154	
		50	0.001475	0.004232	0.007686	0.012345	0.017021	0.021241	0.025934	0.032206	0.039346	0.043154	
		51	0.001475	0.004232	0.007686	0.012345	0.017021	0.021241	0.025934	0.032206	0.039346	0.047988	
	Analytic Method		0.001475	0.004232	0.007686	0.012345	0.017021	0.021241	0.025934	0.032206	0.039347	0.047990	
			10	0.000880	0.002949	0.007612	1.297032	235.8851	16055.02	553491.9	116399.1	16835071	18.16e+8
			12	0.000686	0.002880	0.006036	-0.07952	-41.1843	-5983.41	-392210	-143933.2	-33993185	-56.82e+8
6.0	DTM	30	0.000683	0.002875	0.006060	0.009875	0.014174	0.018670	0.022051	-8.826427	-15889.24	-128318.6	
		40	0.000683	0.002875	0.006060	0.009875	0.014174	0.018676	0.023831	0.029826	0.034546	-16.30851	
		42	0.000683	0.002875	0.006060	0.009875	0.014174	0.018675	0.023835	0.029826	0.036908	-0.539106	
		45	0.000683	0.002875	0.006060	0.009875	0.014174	0.018663	0.023836	0.029806	0.036945	0.048321	
		50	0.000683	0.002875	0.006060	0.009875	0.014174	0.018663	0.023836	0.029806	0.036945	0.045490	
	Analytic Method		0.000682	0.002874	0.006059	0.009870	0.014176	0.018665	0.023830	0.029807	0.036948	0.045489	

Table 2(c) The lateral displacement of the both ends simply supported beam subjected to $P_0 = 9.806$ kN at $\xi = 5$ m.

Table 2(d) The lateral displacement of the both ends simply supported beam subjected to $P_0 = 9.806$ kN at $\xi = 5$ m

Table 3(a) The lateral displacement of one end fixed and other end simply supported beam subjected to $P(x, t) = 29.418 \sin(5t)$ kN/m

x (m)	Method	N	$t = 0.1$ s.	$t = 0.2$ s.	$t = 0.3$ s.	$t = 0.4$ s.	$t = 0.5$ s.	$t = 0.6$ s.	$t = 0.7$ s.	$t = 0.8$ s.	$t = 0.9$ s.	$t = 1.0$ s.
1.0	DTM	5	0.000769	0.004595	0.011689	0.048041	0.484692	4.196953	26.13474	125.1333	490.62539	1647.4996
		8	0.000766	0.004581	0.010886	0.018612	-0.01357	-1.12259	-19.4026	-220.1487	-1843.229	-12171.00
		10	0.000766	0.004568	0.010850	0.018728	0.023598	-0.12344	-4.87652	-97.24722	-1336.588	-13760.01
		20	0.000766	0.004568	0.010850	0.018664	0.026005	0.029410	0.029198	0.026275	-0.006797	-2.614767
		30	0.000766	0.004568	0.010850	0.018664	0.026005	0.029410	0.029198	0.026310	0.022557	0.021032
		36	0.000766	0.004568	0.010850	0.018664	0.026005	0.029410	0.029198	0.026310	0.022557	0.021032
	Analytic Method		0.000765	0.004568	0.010850	0.018664	0.025949	0.029412	0.029021	0.026311	0.022560	0.021032
	DTM	5	0.001503	0.010016	0.027542	0.077313	0.506671	4.031390	24.79243	118.4622	464.3264	1559.236
		8	0.001433	0.009740	0.026531	0.049430	0.145014	-0.69970	-13.2193	-150.7991	-1263.203	-8341.622
		10	0.001430	0.009700	0.026427	0.049416	0.071087	-0.01489	-3.26204	-66.53032	-915.4396	-9425.095
		20	0.001430	0.009700	0.026427	0.049207	0.072430	0.089460	0.097700	0.098562	0.074859	-1.713191
		30	0.001430	0.009700	0.026427	0.049207	0.072430	0.089460	0.097700	0.098354	0.094763	0.092142
		36	0.001430	0.009700	0.026427	0.049207	0.072430	0.089460	0.097700	0.098354	0.094763	0.092142
	Analytic Method		0.001428	0.009704	0.026423	0.049203	0.072433	0.089465	0.097700	0.098357	0.094762	0.092141
2.0	DTM	5	0.001651	0.011782	0.034671	0.084324	0.358456	2.415643	14.38502	68.270403	267.10406	896.33013
		8	0.001257	0.010271	0.032791	0.067892	0.145014	1.234922	18.52561	208.08304	1740.547	11491.563
		10	0.001253	0.010219	0.032637	0.067320	0.109920	0.291637	4.817340	92.184809	1264.1733	13012.35
		20	0.001253	0.010219	0.032637	0.067005	0.107098	0.146316	0.177549	0.197952	0.234568	2.698904
		30	0.001253	0.010219	0.032637	0.067005	0.107098	0.146316	0.177549	0.197170	0.206344	0.206231
		36	0.001253	0.010219	0.032637	0.067005	0.107098	0.146316	0.177549	0.197170	0.206344	0.206231
		Analytic Method		0.001250	0.010219	0.032636	0.067002	0.107099	0.146314	0.177542	0.197170	0.206342
	DTM	5	0.003126	0.017071	0.044365	0.280025	3.331516	29.27856	182.4298	873.1222	3422.074	11487.726
		8	0.001140	0.009439	0.031577	0.070417	0.142671	0.759163	9.945070	110.21784	920.6161	6077.205
		10	0.001133	0.009390	0.031440	0.069850	0.124125	0.261297	2.723425	49.46197	675.9541	6955.7962
		20	0.001133	0.009390	0.031440	0.069845	0.122470	0.183421	0.243111	0.292174	0.336392	1.659259
		30	0.001133	0.009390	0.031440	0.069845	0.122470	0.183421	0.243111	0.291563	0.321190	0.327010
		36	0.001133	0.009390	0.031440	0.069845	0.122470	0.183421	0.243111	0.291563	0.321190	0.327010
	Analytic Method		0.001133	0.009391	0.031443	0.069848	0.122469	0.183423	0.243113	0.291563	0.321190	0.327015

Table 3(b) The lateral displacement of one end fixed and other end simply supported beam subjected to $P(x, t) = 29.418 \cdot \sin(5t)$ kN/m

x (m)	Method	N	$t = 0.1$ s.	$t = 0.2$ s.	$t = 0.3$ s.	$t = 0.4$ s.	$t = 0.5$ s.	$t = 0.6$ s.	$t = 0.7$ s.	$t = 0.8$ s.	$t = 0.9$ s.	$t = 1.0$ s.
5.0	DTM	5	0.011607	0.048979	0.098638	1.264264	18.45872	166.4664	1041.466	4988.7505	19557.18	65658.107
		8	0.001322	0.009514	0.029713	0.067756	0.084250	-1.02579	-20.3944	-233.8909	-1960.564	-12947.57
		10	0.001313	0.009461	0.29645	0.067798	0.123854	0.042280	-4.77837	-99.93325	-1377.559	-14185.19
		20	0.001313	0.009461	0.29645	0.067794	0.126267	0.199931	0.278910	0.349377	0.365221	-2.308988
		30	0.001313	0.009461	0.29645	0.067794	0.126267	0.199931	0.278910	0.349390	0.395487	0.408519
		36	0.001313	0.009461	0.29645	0.067794	0.126267	0.199931	0.278910	0.349390	0.395487	0.408519
	Analytic Method		0.001312	0.009468	0.29642	0.067794	0.126262	0.199838	0.278910	0.349392	0.395489	0.408519
	DTM	5	0.054692	0.214263	0.383837	6.148683	93.26521	844.9059	5289.981	25343.771	99358.36	333574.41
		8	0.001286	0.009592	0.030400	0.068371	0.105888	-0.39791	-9.80029	-113.8592	-955.7575	-6313.221
		10	0.001242	0.009440	0.030297	0.068571	0.125466	0.146926	-1.41304	-33.20760	-460.7263	-4746.778
		20	0.001242	0.009439	0.030297	0.068680	0.126490	0.199946	0.279664	0.351744	0.391071	-0.488667
		30	0.001242	0.009439	0.030297	0.068680	0.126490	0.199946	0.279664	0.351660	0.401417	0.420285
		36	0.001242	0.009439	0.030297	0.068680	0.126490	0.199946	0.279664	0.351660	0.401417	0.420285
	Analytic Method		0.001243	0.009439	0.030297	0.068689	0.126498	0.199949	0.279666	0.351660	0.401417	0.420282
7.0	DTM	5	0.276569	1.064507	1.850677	31.364468	479.5340	4348.105	27227.62	130448.6	511416.98	1716972
		8	0.001372	0.010610	0.033453	0.071749	0.130383	0.391297	3.751191	40.028551	332.91903	2196.184
		10	0.001148	0.009770	0.032756	0.071299	0.125223	0.349121	5.562690	105.7436	1449.227	14916.196
		20	0.001147	0.009759	0.032740	0.071990	0.122958	0.183797	0.244545	0.299006	0.370059	3.218744
		30	0.001147	0.009759	0.032740	0.071990	0.122958	0.183797	0.244545	0.298958	0.338510	0.359374
		36	0.001147	0.009759	0.032740	0.071990	0.122958	0.183797	0.244545	0.298958	0.338510	0.359374
		Analytic Method	0.001143	0.009759	0.032741	0.071291	0.122951	0.183798	0.245458	0.298959	0.338519	0.359373
	DTM	5	1.372021	5.256139	9.07617	156.07416	2390.411	21678.58	135753.8	650404.3	2549875	8560649
		8	0.002595	0.015116	0.036392	0.065692	-0.07046	-5.06331	-87.6715	-995.1664	-8332.646	-55020.98
		10	0.001389	0.010490	0.032059	0.065654	0.105684	0.171700	1.016960	16.80407	228.2705	2347.864
		20	0.001382	0.010590	0.032056	0.065644	0.105659	0.145703	0.180696	0.208948	0.236127	0.709027
		30	0.001382	0.010530	0.032056	0.065644	0.105659	0.145703	0.180695	0.208925	0.231435	0.249105
		36	0.001382	0.010530	0.032056	0.065644	0.105659	0.145703	0.180695	0.208925	0.231435	0.249105
	Analytic Method		0.001384	0.010531	0.032193	0.065643	0.105656	0.145704	0.180695	0.208928	0.231435	0.249105

Table 3(c) The lateral displacement of one end fixed and other end simply supported beam subjected to $P(x, t) = 29.418 \sin(5t)$ kN/m

6. Discussions

In application of DTM, the lateral displacements of one end fixed and the other end simply and both ends simply supported beam are calculated by increasing series size N . In (Tables 1-3), converges of the lateral displacements are introduced. Tables 1 and 2 indicate that change in results of the lateral displacements of both ends simply supported Euler-Bernoulli beam are negligible within the range of $t = 0.1\text{s}-0.2\text{s}$ and series size, N , larger than 12, and within the range of $t = 0.3\text{s}-0.6\text{s}$ and N larger than 30. Besides, Table 3 indicates that change in results of the lateral displacements of one end fixed and other end simply supported beam are negligible within the range of $t = 0.1\text{s}-0.25\text{s}$ and series size N larger than 10, and within the range of $t = 0.3\text{s}-0.7\text{s}$ and N larger than 20, within the range of $t = 0.8\text{s}-1.0\text{s}$ and N larger than 30.

The values of the lateral displacements of the beams using DTM become fixed for the case that the series is taken higher than a definite value. Tables 1, 2, 3 and Figs. 1, 2, 3 are achieved that the results of DTM agree well with the results of analytical solutions.

7. Conclusions

In this study, lateral displacements of one end fixed and the other end simply and both ends simply supported beams are calculated using DTM. Then, the calculated displacements are compared with the result of the analytical solutions. It is seen from the results of DTM and analytical solution that rate of convergence and accuracy of DTM is very good, that the lateral displacements of both ends simply supported and one end fixed other end simply supported beams using analytical method and obtained DTM for $N = 50$ and $N = 30$ overlap, respectively.

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