

Passive suppression of helicopter ground resonance instability by means of a strongly nonlinear absorber

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Abstract. In this paper, we study a problem of passive suppression of helicopter Ground Resonance (GR) using a single degree freedom Nonlinear Energy Sink (NES), GR is a dynamic instability involving the coupling of the blades motion in the rotational plane (i.e. the lag motion) and the helicopter fuselage motion. A reduced linear system reproducing GR instability is used. It is obtained using successively Coleman transformation and binormal transformation. The analysis of the steady-state responses of this model is performed when a NES is attached on the helicopter fuselage. The NES involves an essential cubic restoring force and a linear damping force. The analysis is achieved applying complexification-averaging method. The resulting slow-flow model is finally analyzed using multiple scale approach. Four steady-state responses corresponding to complete suppression, partial suppression through strongly modulated response, partial suppression through periodic response and no suppression of the GR are highlighted. An algorithm based on simple criterions is developed to predict these steady-state response regimes. Numerical simulations of the complete system confirm this analysis of the slow-flow dynamics. A parametric analysis of the influence of the NES damping coefficient and the rotor speed on the response regime is finally proposed.

Keywords: helicopter ground resonance; passive control; nonlinear energy sink; relaxation oscillations; strongly modulated response

1. Introduction

Ground Resonance (GR) is a potential destructive mechanical instability that occurs in helicopters, generally when the aircraft is on the ground. The phenomenon of GR involves a coupling between the airframe motion on its landing gear and the blades motion in the rotational plane (i.e., the lag motion), It can be investigated without taking into account the aerodynamical effects. The standard reference of the GR analysis is the paper by Coleman and Feingold (Coleman and Feingold 1958) where it is established, considering an isotropic rotor, that GR is a flutter instability due to a frequency coalescence between a lag mode and the fuselage mode. The range of rotors speeds Ω for which this frequency coalescence occurs is predicted analytically. More references can be found in (Bramwell *et al.* 2001, Kryszinski and Malburet 2009), a recent analysis of helicopter GR with asymmetric blades is proposed in (Sanches *et al.* 2012), Traditionally, GR

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instability is prevented by two passive methods: increasing the damping (Done 1974) or modify the stiffness of the rotor blade lag mode or the fuselage mode. Active control of GR has been also studied (Krysinski and Malburet 2009),

Targeted Energy Transfer (TET) is a concept based on an additional essentially nonlinear attachment also named Nonlinear Energy Sink (NES) to an existing primary linear system. TET has been extensively studied numerically, theoretically and experimentally, the results prove that the NES is very efficient for vibration mitigation (Vakatis *et al.* 2008) and noise reduction (Bellet *et al.* 2010), Impulsive loading was theoretically analyzed for example in (Vakakis and Gendelman 2001) where TET is investigated in terms of resonance capture. Among a large number of applications, NESs can be used to control dynamic instabilities. In the context of air-craft applications, two companion papers (Lee *et al.* 2007a, Lee *et al.* 2007b) demonstrated that a NES coupled to a rigid wing in subsonic flow can partially or even completely suppress aeroelastic flutter instability. Furthermore, an asymptotic analysis using complexification-averaging method (Manevitch 1999) together with multiple scale method is performed in (Gendelman *et al.* 2010), The paper demonstrates the existence of the three possible passive control mechanisms which are characterized in terms of periodic and strongly modulated responses (Starosvetsky and Gendelman 2008), Finally, suppression of aeroelastic instability of a general nonlinear multi degree of freedom system has also be considered in (Luongo and Zulli 2014), Consequently, in this context, the use of a NES appears to be an interesting alternative passive control of GR instability. Indeed, the only possible effect of using linear dampers is to suppress completely (or not at all) the instability and the adding damping needed for the suppression may be very large. For its part, a NES attachment with a relatively small linear damping and a pure nonlinear stiffness, is able to prevent destructive amplitude of oscillations even if GR instability persists. These situations are hereafter referred as *partial suppression mechanisms* though periodic or strongly modulated responses. The goal of the paper, is therefore to study the effect of attaching a NES on the fuselage on the helicopter GR instability. We focus on the characterization of the steady-state response regimes of a helicopter ground resonance model including a ungrounded NES attachment.

The paper is organized as follows. In Sect. 2, firstly, a reduced helicopter model reproducing GR phenomenon is presented. Then, a NES is attached to the fuselage in an ungrounded configuration leading to the model under study named Helicopter Model including a NES (hereafter referred as HM+NES), The Sect. 3 first describes some the steady-state response regimes of the HM+NES. For regimes are highlighted, they are classified into two categories depending on the fact that the trivial solution of the HM+NES is stable or not. Then, an analytical developments based on complexification-averaging method (Manevitch 1999) together with multiple scale approach (Nayfeh 2011) is developed to analyze situations for which trivial solution of the HM+NES is unstable. Moreover, a systematic procedure for the prediction of the steady-state response regimes is presented. In Sect. 4, the procedure is used to analyze the influence of the damping of the NES and the rotor speed on the response regimes. Finally numerical validation of the method is performed.

2. System under study

2.1 Helicopter Model (HM)

2.1.1 Initial model

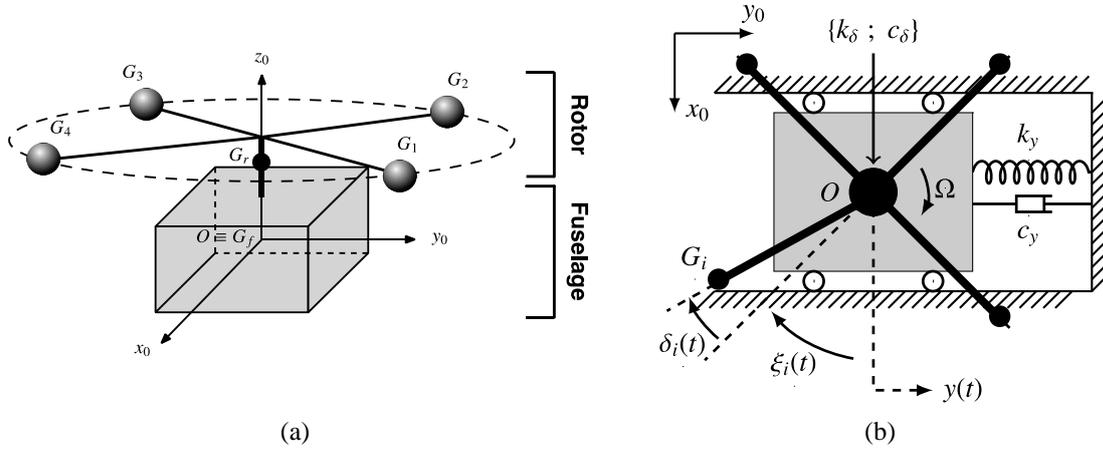


Fig. 1 Descriptive diagram of the used helicopter system. (a) Overview of the system. (b) View from the top

The system under study is shown Fig. 1. It describes an idealized helicopter which consists of a fuselage on which a 4-blades rotor rotates at a constant speed Ω . It is very similar to that described for example in (Johnson 1994, Bramwell *et al.* 2001, Kryszinski and Malburet 2009), The origin O of the earth-fixed system of coordinates (O, x_0, y_0, z_0) coincides with the center of inertia G_f of the fuselage at rest (see Fig. 1). At rest, the center of inertia of the rotor G_r is also located on the axis (O, z_0) . The fuselage is a simple damped mass-spring system with only one translational DOF $y(t)$. Blades, which are assumed to be a mass points G_i (with $i \in [1, 4]$) placed at a distance L from the axis (O, z_0) , are described by the lagging angles $\delta_i(t)$. A lagging angle is the angle between the current position of the blade and its equilibrium position $\xi_i(t) = \Omega t - \frac{\pi}{2}(i-1)$ (see Fig. 1(b)),

The equations of motion which govern the time evolution of the five DOFs of the model (the fuselage displacement $y(t)$ and the four lagging angles $\delta_i(t)$) are derived using Lagrange method. This leads to the reference model

$$(m_y + 4m_\delta)\ddot{y} + c_y\dot{y} + k_y y + M_\delta \sum_{j=1}^4 \left\{ \ddot{\delta}_j \cos(\xi_j + \delta_j) - (\Omega + \dot{\delta}_j)^2 \sin(\xi_j + \delta_j) \right\} = 0 \quad (1)$$

$$I_\delta \ddot{\delta}_i + c_\delta \dot{\delta}_i + k_\delta \delta_i + M_\delta \ddot{y} \cos(\xi_i + \delta_i) = 0, \quad i = 1, 4 \quad (2)$$

where “ \cdot ” denotes the derivative with respect to time t , m_y is the fuselage mass, m_δ is the mass of a blade, $M_\delta = m_\delta L$ and $I_\delta = m_\delta L^2$ are the static moment and the moment of inertia of one blade respectively, c_y , c_δ are damping coefficients of the the fuselage and of a blade respectively and k_y and k_δ are linear stiffness coefficients.

After linearization of the reference model (1-2) around the trivial equilibrium point, a change of variables which transforms individual motions of the blades (described by the lagging angles) into collective motions described by the so-called Coleman coordinates (Coleman and Feingold 1958) is applied. For a 4-blades rotor there are four Coleman coordinates δ_0 , δ_{1c} , δ_{1s} and δ_{cp} defined by

$$\delta_0(t) = \frac{1}{4} \sum_{j=1}^4 \delta_j(t), \delta_{1c}(t) = \frac{1}{2} \sum_{j=1}^4 \delta_j(t) \cos(\xi_j(t)), \quad (3)$$

$$\delta_{1s}(t) = \frac{1}{2} \sum_{j=1}^4 \delta_j(t) \sin(\xi_j(t)), \delta_{cp}(t) = \frac{1}{4} \sum_{j=1}^4 (-1)^j \delta_j(t). \quad (4)$$

One can be shown that the variables δ_0 and δ_{cp} are uncoupled and can be discarded. The reason of the decoupling is the fact that the collective motions δ_0 and δ_{cp} leave the rotor center of inertia motionless. As a result, a system of equations with three DOF, namely y , δ_{1c} and δ_{1s} , is obtained.

Introducing the following notations

$$\begin{aligned} \omega_y^2 &= k_y / (m_y + 4m_\delta), & \omega_\delta^2 &= k_\delta / I_\delta; \\ \tilde{\lambda}_y &= c_y / (m_y + 4m_\delta), & \tilde{\lambda}_\delta &= c_\delta / I_\delta \\ \tilde{S}_d &= 2M_\delta / (m_y + 4m_\delta), & \tilde{S}_c &= M_\delta / I_\delta = 1/L. \end{aligned} \quad (5)$$

where ω_y and ω_δ are the natural frequency of the fuselage and of one blade respectively and \tilde{S}_c and \tilde{S}_d are the coupling coefficients, equations of motion are finally written in matrix form

$$MX + (C + G)X + KX = 0, \quad \text{with} \quad X = [y \ \delta_{1c} \ \delta_{1s}]^T. \quad (6)$$

M , K , C and G , are mass matrix, stiffness matrix, damping matrix and gyroscopic matrix of the system respectively, they are defined by

$$M = \begin{bmatrix} 1 & \tilde{S}_d & 0 \\ \tilde{S}_c & 1 & 0 \\ 0 & 0 & 1 \end{bmatrix}, K = \begin{bmatrix} \omega_y^2 & 0 & 0 \\ 0 & \omega_\delta^2 - \Omega^2 & \tilde{\lambda}_\delta \Omega \\ 0 & -\tilde{\lambda}_\delta \Omega & \omega_\delta^2 - \Omega^2 \end{bmatrix}, \quad (7)$$

$$C = \begin{bmatrix} \tilde{\lambda}_y & 0 & 0 \\ 0 & \tilde{\lambda}_\delta & 0 \\ 0 & 0 & \tilde{\lambda}_\delta \end{bmatrix}, G = \begin{bmatrix} 0 & 0 & 0 \\ 0 & 0 & 2\Omega \\ 0 & -2\Omega & 0 \end{bmatrix}. \quad (8)$$

Note that \tilde{S}_d and \tilde{S}_c characterize the fuselage/rotor coupling.

2.1.2 Reduced model

The reduced model is obtained retaining only the DOF that occurs in helicopter GR. The GR phenomenon is explained making a stability analysis of the previous linear system (6), The set of eigenvalues α_i (with $i \in [1,6]$) are easily computed from M , K , C and G . Note that if the fuselage/rotor coupling is suppressed (i.e., stating $\tilde{S}_c = \tilde{S}_d = 0$), the eigenvalues of the system correspond to the natural eigenvalues of the fuselage, denoted $\alpha_{f,i}$ (with $i \in [1,2]$), and of the rotor, denoted $\alpha_{r,i}$ (with $i \in [1,4]$), In Fig. 2(a), the typical behavior of the imaginary part of these eigenvalues is reported with respect to the rotor speed Ω for $\omega_y < \omega_\delta$. We can notice that there are two values of Ω for which an interaction between the fuselage mode and the regressive rotor mode

is possible: $\Omega \approx |\omega_y - \omega_\delta|$ and $\Omega \approx \omega_y + \omega_\delta$ ¹.

In Figs. 2(b) and 2(c), the comparison between eigenvalues of the uncoupled systems $\alpha_{f,i}$ and $\alpha_{r,i}$ and the eigenvalues α_i of the coupled system shows that:

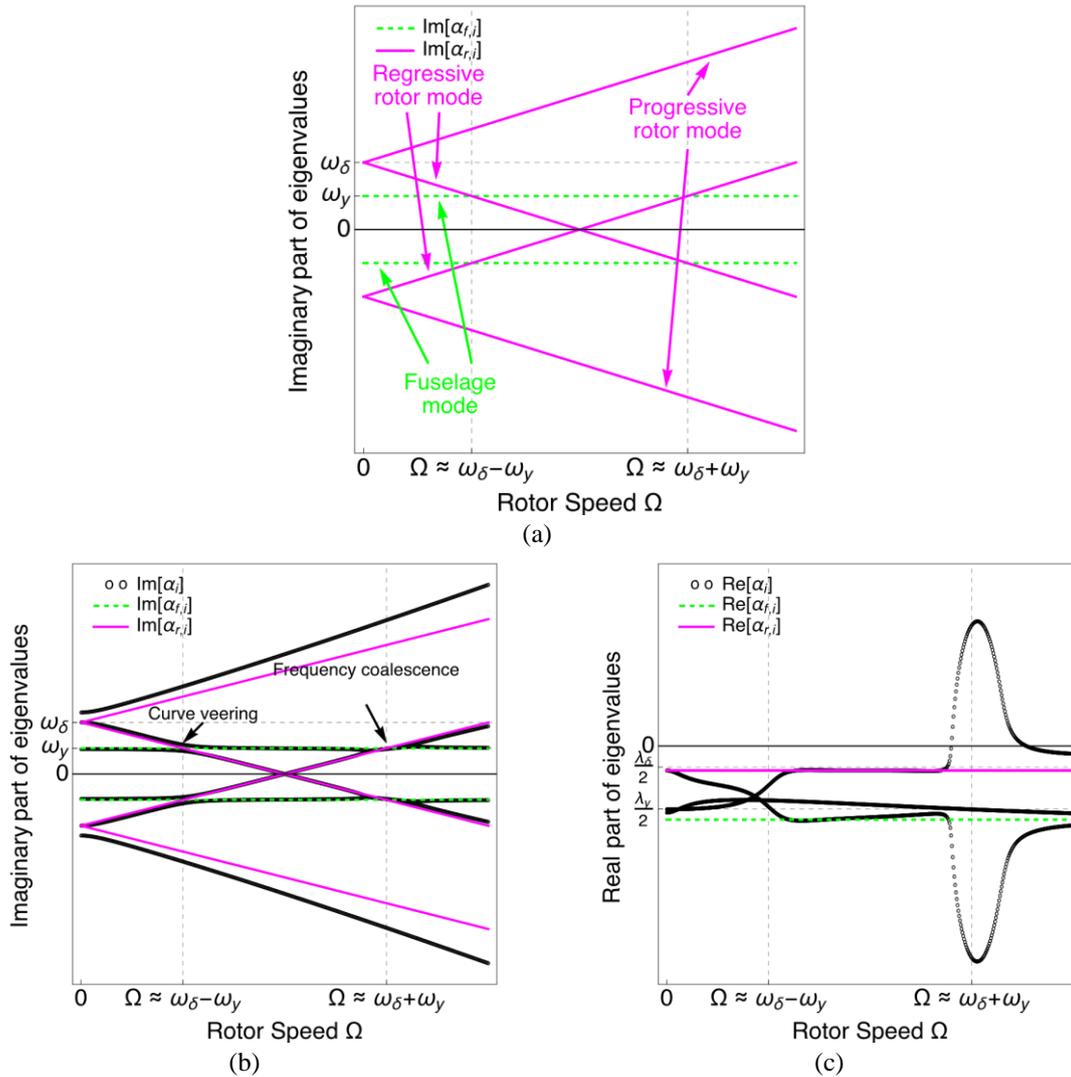


Fig. 2 Eigenvalues of the uncoupled system and coupled system (6) for $\omega_y < \omega_\delta$. (a) Imaginary parts of the natural eigenvalues α_f of the fuselage (dashed blue line) and α_r of the rotor (solid red line), Comparison between the eigenvalues of the uncoupled systems (i.e. α_f and α_r) and the eigenvalues α of the coupled system (black circles), (b) imaginary parts and (c) real parts. Parameters used: $\omega_y=1$, $\omega_\delta=2$, $\tilde{\lambda}_y=0.105$, $\tilde{\lambda}_\delta=0.035$, $\tilde{S}_c=0.7$ and $\tilde{S}_d=0.35$. The parameters $\tilde{\lambda}_y$, $\tilde{\lambda}_\delta$, \tilde{S}_c and \tilde{S}_d are chosen to obtain readable figures, no for their realism

¹If undamped system would be considered (i.e., if $\lambda_y = \lambda_\delta = 0$), we would get exact equalities: $\Omega \approx |\omega_y - \omega_\delta|$ and $\Omega \approx \omega_y + \omega_\delta$, see (Johnson 1994) for more details

- For $\Omega \approx |\omega_y - \omega_\delta|$, a phenomenon of “curve veering” (Leissa 1974) appears, the real part of the eigenvalues α stay negative and there is no instability.
- For $\Omega \approx \omega_y + \omega_\delta$, a phenomenon of frequency coalescence is observed, the real part of one of the eigenvalues α becomes positive and a dynamic instability occurs; this is the helicopter ground resonance.

Fig. 2 shows also that the progressive rotor mode does not interact with the fuselage mode. For $\omega_y > \omega_\delta$, similar observations can be made and consequently, in both situations ($\omega_y < \omega_\delta$ and $\omega_y > \omega_\delta$), the progressive rotor mode does not contribute to the creation of the GR instability. Therefore, the last step to obtain the simplest model for helicopter ground resonance is to eliminate the progressive rotor mode from the equations of motion. This is achieved in the remaining of the section applying bi-normal transformation (Caughey and O’Kelly 1963, Done 1974) to the equations of motion of the rotor alone. In the state-space form we obtain

$$U_r = A_r U_r, \quad \text{with} \quad U_r = \begin{bmatrix} \delta_{1c} & \delta_{1s} & \dot{\delta}_{1c} & \dot{\delta}_{1s} \end{bmatrix}. \quad (9)$$

The following eigenvalue problems

$$A_r r = \alpha_r r \quad \text{and} \quad A_r^t l = \alpha_r l \quad (10)$$

where A_r^t denotes the transpose of A_r , are solved giving:

- two pairs of complex conjugates eigenvalues: $\alpha_{r,1}$, $\alpha_{r,1}^*$, $\alpha_{r,2}$ and $\alpha_{r,2}^*$ (the “*” is the usual notation for the complex conjugate),
- two pairs of complex conjugates eigenvectors of A_r , r_i , called *right eigenvectors* of A_r : r_1 , r_1^* , r_2 and r_2^* .
- two pairs of complex conjugates eigenvectors of A_r^t , l_i , called *left eigenvectors* of A_r : l_1 , l_1^* , l_2 and l_2^* .

The right and left eigenvectors satisfy the biorthogonality properties: $L^t R$ and $L^t A_r R$ are diagonal matrices where $R = [r_1 \ r_1^* \ r_2 \ r_2^*]$ and $L = [l_1 \ l_1^* \ l_2 \ l_2^*]$. It is convenient to normalize the two set of eigenvectors r_i and l_i in order to obtain

$$L^t R = I. \quad (11)$$

In this case, we have

$$L^t A_r R = \begin{bmatrix} \alpha_{r,1} & 0 & 0 & 0 \\ 0 & \alpha_{r,1}^* & 0 & 0 \\ 0 & 0 & \alpha_{r,2} & 0 \\ 0 & 0 & 0 & \alpha_{r,2}^* \end{bmatrix} = D_r. \quad (12)$$

The binormal transformation consists in introducing the binormal coordinates which are constituted of two pairs of complex conjugates, (q_1, q_1^*) and (q_2, q_2^*) , and defined by the following relation

$$U_r = RQ \Leftrightarrow Q = L^t U_r, \quad \text{with} \quad Q = [q_1 \ q_1^* \ q_2 \ q_2^*]. \quad (13)$$

Introducing Eq. (13) in Eq. (9), the equations of motion of the rotor take the form of the

following diagonal system

$$\dot{Q} = D_r Q. \quad (14)$$

One of the couples (q_1, q_1^*) and (q_2, q_2^*) is relative to the progressive rotor mode and the other the regressive one. The couple (q_2, q_2^*) is arbitrary chosen to be relative to the progressive rotor mode and since this mode does not destabilize the system, the variables q_2 and q_2^* can be removed from the analysis.

Consequently, using the vector U_r , equations of motion of the whole coupled system, i.e., Eq. (6), become

$$\ddot{y} + \tilde{\lambda}_y \dot{y} + \omega_y^2 y + \tilde{S}_d \dot{U}_{r,3} = 0 \quad (15)$$

$$U_r - A_r U_r + \begin{bmatrix} 0 \\ 0 \\ \tilde{S}_c \ddot{y} \\ 0 \end{bmatrix} = 0. \quad (16)$$

Then using Eq. (14) (ignoring variables q_2 and q_2^*) and the following relations

$$\alpha_{r,2} = \alpha_{r,1}^*, R_{32} = R_{31}^*, L_{32} = L_{31}^*, \quad (17)$$

Eqs. (15)-(16) become finally

$$\ddot{y} + \tilde{\lambda}_y \dot{y} + \omega_y^2 y + \tilde{S}_d (R_{31} \dot{q}_1 + R_{31}^* \dot{q}_1^*) = 0 \quad (18)$$

$$\dot{q}_1 - \alpha_{r,1} q_1 + \tilde{S}_c L_{31} \ddot{y} = 0. \quad (19)$$

Eqs. (18)-(19) define the simplest Helicopter Model (the HM with 4 DOF in state-space) that can reproduce GR phenomenon.

2.2 Helicopter Model including a Non Linear Energy Sink (HM+NES)

A NES with a mass m_h , a damping coefficient c_h and a cubic stiffness k_{3h} , is attached on the fuselage in an ungrounded configuration (see Fig. 3), Taking into account the NES displacement $h(t)$, the equations of motion read

$$\ddot{y} + \tilde{\lambda}_y \dot{y} + \omega_y^2 y + \tilde{S}_d (R_{31} \dot{q}_1 + R_{31}^* \dot{q}_1^*) + \tilde{\mu}(\dot{y} - \dot{h}) + \tilde{\alpha}_3 (y - h)^3 = 0 \quad (20)$$

$$\varepsilon \ddot{h} + \tilde{\mu}(\dot{h} - \dot{y}) + \tilde{\alpha}_3 (h - y)^3 = 0 \quad (21)$$

$$\dot{q}_1 - \alpha_{r,1} q_1 + \tilde{S}_c L_{31} \ddot{y} = 0, \quad (22)$$

where $\varepsilon = m_h / (m_y + 4m_\delta)$ is the mass ratio, $\tilde{\mu} = c_h / (m_y + 4m_\delta)$ and $\tilde{\alpha}_3 = k_{3h} / (m_y + 4m_\delta)$.

System of Eqs. (20)-(22) is the reduced Helicopter Model including a Non Linear Energy Sink (HM+NES),

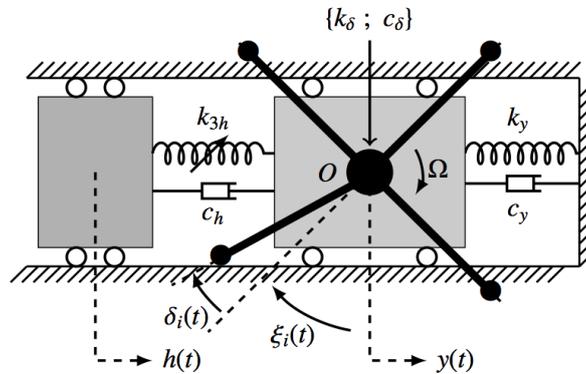


Fig. 3 Descriptive diagram of the used helicopter system coupled to an ungrounded NES. View from the top

It is easy to show that the only fixed point of the HM+NES (Eqs. (20)-(22)) is the trivial solution $y=h=q_1=0$. To find its stability, the 6 eigenvalues of the Jacobian matrix of the state-space representation of the system Eqs. (20)-(22) have to be computed. The trivial solution is unstable if one of the eigenvalues has a positive real part.

3. Steady-state response regimes of the HM+NES

3.1 Some steady-state response regimes

Using numerical integration of the HM+NES, Eqs. (20)-(22), four different types of response regimes which may be highlighted (as illustrated in Sect. 4) when a NES is attached on the fuselage. They are classified into two categories depending on the fact that the trivial solution of the HM+NES is stable or not:

- **The trivial solution of the HM+NES is stable:**

- *Complete suppression.* In this case, the additional damping due to the NES attachment stabilizes the system and the GR instability is completely suppressed.

- **The trivial solution of the HM+NES is unstable:**

- *Partial suppression through Periodic Response (PR),* In this case, the steady-state response regime is periodic with frequency close to ω_y^2 .

- *Partial suppression through Strongly Modulated Response (SMR),* In this case, the steady-state response regime is a quasiperiodic regime which exhibits a “fast” component with frequency close to ω_y and a “slow” component corresponding to the envelope of the signal. The term “Strongly modulated response” has been introduced by Starosvetsky and Gendelman (2008) for the study of a forced linear system coupled to a NES.

- *No suppression of GR.* The NES is not able to maintain stable steady-state regimes. We observe exponential growth of the fuselage displacement.

These four responses are also observed by Lee *et al.* (2007a) and study theoretically by Gendelman *et al.* (2010) in the context of the control aeroelastic instabilities of a rigid wing in

²This can be shown for example by computing the power spectrum of the steady part of the signal.

subsonic flow by means of a NES.

In the following sections an analytical procedure based on complexification-averaging method together with multiple scale method is developed to analyze situations for which trivial solution of the HM+NES is unstable.

3.2 The complexified-averaged model

The analytical study proposed is based on *complexification-averaging method* first introduced by Manevitch [Manevitch, 1999] and discussed in detail by Vakakis *et al.* (2008),

First, to simplify the following calculations, it is convenient to introduce barycentric coordinates $v(t)$ and $w(t)$

$$v = y + \varepsilon h \text{ and } w = y - h. \quad (23)$$

Using Eqs. (23), Eqs. (20)-(22) are written as follow

$$\ddot{v} + \tilde{\lambda}_y \frac{\dot{v} + \varepsilon \dot{w}}{\varepsilon + 1} + \omega_y^2 \frac{v + \varepsilon w}{\varepsilon + 1} + \tilde{S}_d (R_{31} \dot{q}_1 + R_{31}^* \dot{q}_1^*) = 0 \quad (24)$$

$$\begin{aligned} \ddot{w} + \tilde{\lambda}_y \frac{\dot{v} + \varepsilon \dot{w}}{\varepsilon + 1} + \omega_y^2 \frac{v + \varepsilon w}{\varepsilon + 1} + \tilde{S}_d (R_{31} \dot{q}_1 + R_{31}^* \dot{q}_1^*) + \\ \tilde{\mu} \frac{1 + \varepsilon}{\varepsilon} \dot{w} + \tilde{\alpha}_3 \frac{1 + \varepsilon}{\varepsilon} w^3 = 0 \end{aligned} \quad (25)$$

$$\dot{q}_1 - \alpha_{r,1} q_1 + \tilde{S}_c L_{31} \frac{\ddot{v} + \varepsilon \ddot{w}}{\varepsilon + 1} = 0. \quad (26)$$

Secondly, the complexification³ consists in introducing the following change of variable

$$\psi_1 = \dot{v} + j\omega_y v \text{ and } \psi_2 = \dot{w} + j\omega_y w, \quad (27)$$

with $j^2 = -1$.

Previous numerical and theoretical results (see Sect. 3.1) motive us to introduce the assumption that the variable v , w and q_1 may be broken down into fast and slow components. For that, the following representation is introduced

$$\psi_1 = \phi_1 e^{j\omega_y t}, \psi_2 = \phi_2 e^{j\omega_y t}, q_1 = \phi_3 e^{j\omega_y t}, \quad (28)$$

where ϕ_i (with $i \in [1,3]$) is the complex slow modulated amplitude of the fast component $e^{j\omega_y t}$.

Substituting Eqs. (27) into Eqs. (24-26) an equivalent complex system of differential equations is obtained. Substituting next Eq. (28) in this complex system and performing an averaging over one period of the frequency ω_y yield to a system of equations describing the behavior of the slow complex amplitudes ϕ_i

³This step is not necessary for the variable $q_1(t)$ because it is already a complex variable.

$$\dot{\phi}_1 + j \frac{\omega_y}{2} \phi_1 + \frac{\tilde{\lambda}_y - j\omega_y}{2(1+\varepsilon)} (\phi_1 + \varepsilon\phi_2) + \tilde{S}_d R_{31} (\dot{\phi}_3 + j\omega_y \phi_3) = 0 \quad (29)$$

$$\begin{aligned} \dot{\phi}_2 + j \frac{\omega_y}{2} \phi_2 + \frac{\tilde{\lambda}_y - j\omega_y}{2(1+\varepsilon)} (\phi_1 + \varepsilon\phi_2) + \tilde{S}_d R_{31} (\dot{\phi}_3 + j\omega_y \phi_3) + \\ \frac{\tilde{\mu}}{2} \frac{1+\varepsilon}{\varepsilon} \phi_2 - j \frac{3\tilde{\alpha}_3}{8\omega_y^3} \frac{1+\varepsilon}{\varepsilon} \phi_2 |\phi_2|^2 = 0 \end{aligned} \quad (30)$$

$$\dot{\phi}_3 + (j\omega_y - \alpha_{r,1}) \phi_3 + \frac{\tilde{S}_c}{1+\varepsilon} L_{31} \left[(\dot{\phi}_1 + \varepsilon\dot{\phi}_2) + j \frac{\omega_y}{2} (\phi_1 + \varepsilon\phi_2) \right] = 0. \quad (31)$$

Eqs. (29)-(31) define the complexified-averaged system.

3.3 Approximation of the periodic solutions of the HM+NES and their stability

Fixed points of the complexified-averaged system (29)-(31) (defined as $\dot{\phi}_i = 0$ for $i \in [1,3]$) characterize periodic solutions of Eqs. (24)-(26) if the frequency of the periodic solutions is exactly equal to the frequency used to define the complex variables (27), (i.e., the natural frequency of the fuselage ω_y). However, introducing polar form for the variables $\phi_i(t)$

$$\phi_i(t) = n_i(t) e^{j\theta_i(t)}, \quad \text{with } i \in [1,3] \quad (32)$$

and considering not the arguments differences $\delta_{i1} = \theta_i(t) - \theta_1(t)$, the periodic solutions of the system of Eqs. (24)-(26) (and consequently of the HM+NES (20)-(22)) may be defined from the complexified-averaged system (29)-(31) as the fixed points of the system of the equations of motion describing the dynamic of the variables $n_1, n_2, n_3, \delta_{21}$ and δ_{31} . This system is obtain by first rewriting Eqs. (29)-(31) using matrix form

$$\Phi = C\Phi + \phi_2 |\phi_2|^2 H, \quad (33)$$

where the constant complex matrix C and vector H are easy obtained from Eqs. (29)-(31),

Next introducing the polar coordinates (32) and separating real and imaginary parts of each equation, Eq. (33) take the form

$$\dot{n}_1 = n_1 \text{Re}[C_{11}] + n_2 \text{Re}[C_{12} e^{j\delta_{21}}] + n_3 \text{Re}[C_{13} e^{j\delta_{31}}] + n_2^3 \text{Re}[H_1 e^{j\delta_{21}}] \quad (34)$$

$$n_1 \dot{\theta}_1 = n_1 \text{Im}[C_{11}] + n_2 \text{Im}[C_{12} e^{j\delta_{21}}] + n_3 \text{Im}[C_{13} e^{j\delta_{31}}] + n_2^3 \text{Im}[H_1 e^{j\delta_{21}}] \quad (35)$$

$$\dot{n}_2 = n_1 \text{Re}[C_{21} e^{-j\delta_{21}}] + n_2 \text{Re}[C_{22}] + n_3 \text{Re}[C_{23} e^{j(\delta_{31} - \delta_{21})}] + n_2^3 \text{Re}[H_2] \quad (36)$$

$$n_2 \dot{\theta}_2 = n_1 \text{Im}[C_{21} e^{-j\delta_{21}}] + n_2 \text{Im}[C_{22}] + n_3 \text{Im}[C_{23} e^{j(\delta_{31} - \delta_{21})}] + n_2^3 \text{Im}[H_2] \quad (37)$$

$$\dot{n}_3 = n_1 \text{Re}[C_{31} e^{-j\delta_{31}}] + n_2 \text{Re}[C_{32} e^{j(\delta_{21} - \delta_{31})}] + n_3 \text{Re}[C_{33}] + n_2^3 \text{Re}[H_3 e^{j(\delta_{21} - \delta_{31})}] \quad (38)$$

$$n_3 \dot{\theta}_3 = n_1 \text{Im}[C_{31} e^{-j\delta_{31}}] + n_2 \text{Im}[C_{32} e^{j(\delta_{21} - \delta_{31})}] + n_3 \text{Im}[C_{33}] + n_2^3 \text{Im}[H_3 e^{j(\delta_{21} - \delta_{31})}] \quad (39)$$

Note that the right hand sides of Eqs. (34)-(39) do not depend on θ_i but on δ_{21} and δ_{31} .

Finally, combining Eqs. (35) and (37) as $(n_1(37) - n_2(37)(35))/n_1n_2$ and Eqs. (35) and (39) as $(n_1(39) - n_3(39)(35))/n_1n_3$ and grouping with Eqs. (34), (36) and (38), we obtained the close form differential equations

$$\Lambda = F(\Lambda), \quad \text{with} \quad \Lambda = [n_1 \ n_2 \ n_3 \ \delta_{21} \ \delta_{31}]^T. \quad (40)$$

Fixed points $\Lambda^e = [n_1^e \ n_2^e \ n_3^e \ \delta_{21}^e \ \delta_{31}^e]$ of Eqs. (40) are computed by solving $F(\Lambda^e) = 0$ and associated stability property are found by looking the sign of the real parts of the eigenvalues λ_i (with $i \in [1,5]$) of the Jacobian matrix of F evaluated at Λ^e .

This analysis permits to predict the existence of stable periodic response regimes which correspond to the case where the real parts of all the eigenvalues are negative. In the following section, an asymptotic analysis of the complexified-averaged model is developed to characterize response regimes when stable property is not satisfied.

3.4 Asymptotic analysis of the complexified-averaged model

In this section we assume that $\varepsilon=1$ (i.e., the mass of the NES is small with respect to the total mass of the fuselage and the blades) and that the parameters $\tilde{\lambda}_y, \tilde{\lambda}_\delta, \tilde{S}_d, \tilde{S}_c, \tilde{\mu}$ and $\tilde{\alpha}_3$ are of order ε (i.e., $\tilde{\lambda}_y, \tilde{\lambda}_\delta, \tilde{S}_d, \tilde{S}_c, \tilde{\mu}, \tilde{\alpha}_3 : O(\varepsilon)$), These parameters are rescaled as

$$\lambda_y = \frac{\tilde{\lambda}_y}{\varepsilon}; S_d = \frac{\tilde{S}_d}{\varepsilon}; \mu = \frac{\tilde{\mu}}{\varepsilon}; \quad (41)$$

$$\lambda_\delta = \frac{\tilde{\lambda}_\delta}{\varepsilon}; S_c = \frac{\tilde{S}_c}{\varepsilon}; \alpha_3 = \frac{\tilde{\alpha}_3}{\varepsilon}. \quad (42)$$

with $\lambda_y, \lambda_\delta, S_d, S_c, \mu, \alpha_3 : O(1)$.

Moreover, we focus the analysis for Ω around $\omega_y + \omega_\delta$ introducing the detuning term a , defined as

$$\Omega = \omega_y + \omega_\delta + a\varepsilon, \quad (43)$$

with $a:O(1)$,

Using the rescaled parameters, the terms R_{31} , L_{31} and $\alpha_{r,1}$ can be expanded in a first-order Taylor series around $\varepsilon=0$ giving

$$R_{31} = j + O(\varepsilon^2) \quad (44)$$

$$L_{31} = -\frac{\lambda_\delta}{8\omega_\delta} \varepsilon + j \left(\frac{\omega_y + a\varepsilon}{4\omega_\delta} \right) + O(\varepsilon^2) \quad (45)$$

$$\alpha_{r,1} = -\frac{\lambda_\delta}{2} \varepsilon + j(\omega_y + a\varepsilon) + O(\varepsilon^2). \quad (46)$$

Introducing Eqs. (44), (45) and (46) (neglecting the $O(\varepsilon^2)$ terms) and the rescaling parameters (41)-(42), Eqs. (29)-(31) becomes

$$\dot{\phi}_1 + \varepsilon \left(\frac{\lambda_y + j\omega_y}{2} \phi_1 - j \frac{\omega_y}{2} \phi_2 - S_d \omega_y \phi_3 \right) = 0 \quad (47)$$

$$\begin{aligned} \dot{\phi}_2 + \frac{\varepsilon \lambda_y}{2} \phi_1 - j \frac{\omega_y}{2} (\phi_1 - \phi_2)(1 - \varepsilon) - + \\ \varepsilon S_d \omega_y \phi_3 + \phi_2 (1 + \varepsilon) \left(\frac{\mu}{2} - j \frac{3\alpha_3}{8\omega_y^3} |\phi_2|^2 \right) = 0 \end{aligned} \quad (48)$$

$$\dot{\phi}_3 + \varepsilon \left(\left(\frac{\lambda_\delta}{2} - ja \right) \phi_3 - \frac{S_c \omega_y^2}{8\omega_\delta} \phi_1 \right) = 0. \quad (49)$$

Eqs. (47)-(49) define a simplified version of the complexified-averaged system which is called *full slow-flow system* which is now be analyzed using the multiple scale approach (Nayfeh 2011) with respect to the small parameter ε by considering *slow* time $t_0=t$ and *super-slow* time⁴ $t_1=\varepsilon t$. To achieve this, multiple scale expansion is introduced as

$$\phi_i = \phi_i(t_0, t_1) = n_i(t_0, t_1) e^{i\theta_i(t_0, t_1)} \quad \text{for } i = 1, 2, 3, \quad (50)$$

reported in Eqs. (47)-(49) with the derivative rule

$$\frac{d}{dt} = \frac{\partial}{\partial t_0} + \varepsilon \frac{\partial}{\partial t_1}. \quad (51)$$

and finally isolating expressions of order ε^0 and ε^1 are deduced.

3.4.1 Slow time scale: ε^0 -order of the system and Slow Invariant Manifold

The expressions of order ε^0 correspond to the following differential equations

$$\frac{\partial \phi_1}{\partial t_0} = 0 \quad (52)$$

$$\frac{\partial \phi_2}{\partial t_0} = j \frac{\omega_y}{2} (\phi_1 - \phi_2) - \frac{\mu}{2} \phi_2 + j \frac{3\alpha_3}{8\omega_y^3} \phi_2 |\phi_2|^2 \quad (53)$$

$$\frac{\partial \phi_3}{\partial t_0} = 0. \quad (54)$$

At the slow time scale, ϕ_3 is uncoupled with ϕ_1 and ϕ_2 , and Eqs. (52) and (54) can be solved giving

$$\frac{\partial \phi_1}{\partial t_0} = 0 \Rightarrow \phi_1(t_0, t_1) = \Phi_1(t_1) \quad (55)$$

⁴We use the terminology introduced by Gendelman and Bar (2010) whereby the term *fast* is reserved for the fast component $e^{j\omega_y t}$.

$$\frac{\partial \phi_3}{\partial t_0} = 0 \Rightarrow \phi_3(t_0, t_1) = \Phi_3(t_1). \tag{56}$$

Introducing the polar coordinates

$$\phi_j = n_j(t_0, t_1) e^{i\theta_j(t_0, t_1)} \text{ for } j = 1, 2 \text{ and } 3, \tag{57}$$

Eqs. (52)-(53) reduce to

$$\frac{\partial n_1}{\partial t_0} = 0 \tag{58}$$

$$\frac{\partial n_2}{\partial t_0} = \frac{\omega_y}{2} (n_1 \sin \delta_{21} + n_2 F_I(n_2)) \tag{59}$$

$$\frac{\partial n_3}{\partial t_0} = 0 \tag{60}$$

$$\frac{\partial \delta_{21}}{\partial t_0} = \frac{\omega_y}{2} \left(\frac{n_1}{n_2} \cos \delta_{21} - F_R(n_2) \right) \tag{61}$$

$$\frac{\partial \delta_{31}}{\partial t_0} = 0, \tag{62}$$

where (as in Section. (3.3)) the argument difference $\delta_{21} = \theta_2 - \theta_1$ and $\delta_{31} = \theta_3 - \theta_1$ have been introduced and the real scalar functions F_R and F_I are defined as

$$F(n_2) = F_R(n_2) + jF_I(n_2) = 1 - \frac{3\alpha_3}{4\omega_y^4} n_2^2 - j \frac{\mu}{\omega_y}. \tag{63}$$

The fixed points of Eqs. (58)-(62) (which characterize the periodic solutions of Eqs. (24)-(26) at the slow time scale) can be computed directly by solving

$$\lim_{t_0 \rightarrow \infty} \frac{\partial n_2}{\partial t_0} = 0 \text{ and } \lim_{t_0 \rightarrow \infty} \frac{\partial \delta_{21}}{\partial t_0} = 0. \tag{64}$$

Using Eq. (55), Eqs. (61) and (60) reduce to the following form

$$\Phi_1(t_1) = \Phi_2(t_1) F(|\Phi_2(t_1)|), \forall \Phi_3, \tag{65}$$

where

$$\lim_{t_0 \rightarrow \infty} \phi_2(t_0, t_1) = \Phi_2(t_1). \tag{66}$$

It is interesting to note that Eq. (65) corresponds to the fixed point of Eq. (53) i.e.

$$\lim_{t_0 \rightarrow \infty} \frac{\partial \phi_2}{\partial t_0} = 0.$$

As discussed for example in (Starosvetsky and Gendelman 2008) for forced linear system with

NES and in (Gendelman *et al.* 2010, Gendelman and Bar 2010) for nonlinear self-excited system with NES, Eq. (65) defines the so-called *Slow Invariant Manifold* (SIM) but with respect to the variable (Φ_1, Φ_2, Φ_3) ,

Considering the polar coordinates

$$\Phi_j = N_j(t_1) e^{i\Theta_j(t_1)} \text{ for } j=1,2 \text{ and } 3, \quad (67)$$

it is convenient to characterize the SIM on the (N_1, N_2, N_3) -space as

$$N_1^2 = H(N_2) \quad (68)$$

where the real scalar function H is defined by

$$H(N_2) = N_2^2 |F(N_2)|^2 = N_2^2 [F_R(N_2)^2 + F_I(N_2)^2] \quad (69)$$

The SIM does not depend on the variable N_3 and this structure can be analysed in the (N_2, N_1) -plan. The local extrema of the function $H(x)$ are given by the positive roots of its derivative H' . An easy calculus shows that the local extrema occur at

$$N_{2,M} = \frac{2}{3} \omega_y^2 \sqrt{\frac{2 - \sqrt{1 - 3 \frac{\mu^2}{\omega_y^2}}}{\alpha_3}} \quad N_{2,m} = \frac{2}{3} \omega_y^2 \sqrt{\frac{2 + \sqrt{1 - 3 \frac{\mu^2}{\omega_y^2}}}{\alpha_3}} \quad (70)$$

if the following relation holds

$$\mu < \frac{1}{\sqrt{3}} \omega_y \quad (71)$$

because in this case $N_{2,M} < N_{2,m}$.

Stability of the Slow Invariant Manifold. Stability range of the SIM is determined by examining the sign of the eigenvalues real parts of the Jacobian matrix of differential system (59-61) on the SIM. It can be shown that the condition of stability of the SIM is equivalent to

$$H'(N_2) > 0 \quad (72)$$

and the stability range of the CM is characterized by the points $(N_{2,m}, N_{1,m})$ and $(N_{2,M}, N_{1,M})$ where $N_{1,M} = \sqrt{H(N_{2,M})}$, $N_{1,m} = \sqrt{H(N_{2,m})}$, which are therefore called *fold points* (Seydel 2010), A typical *Slow Invariant Manifold* and its stability range are depicted (see Fig. 4(a)) in (N_2, N_1) -plan where $N_{2,d}$ and $N_{2,u}$ are solutions of

$$H(N_{2,m}) = H(N_{2,d}) \Rightarrow N_{2,d} = \frac{2\sqrt{2}}{3} \omega_y^2 \sqrt{\frac{1 - \sqrt{1 - 3 \frac{\mu^2}{\omega_y^2}}}{\alpha_3}}, \quad (73)$$

and

$$H(N_{2,M}) = H(N_{2,u}) \Rightarrow N_{2,u} = \frac{2\sqrt{2}}{3} \omega_y^2 \sqrt{\frac{1 + \sqrt{1 - 3\frac{\mu^2}{\omega_y^2}}}{\alpha_3}}, \tag{74}$$

respectively. In the (N_1, N_2, N_3) -space, each fold point defines a folded line (L_M and L_m) co-linear to the N_3 -axis (see Fig. 4(b)),

The shape and the stability property of the SIM (i.e., the existence of folded lines on which the stability of the SIM changes) shown in Fig. 4 allow to define three steady-state regimes of the full slow-flow system (47-49) that can explain the three steady-state regimes of the HM+NES described in Sect. 3.1.

These situations corresponds to a periodic solution of the HM+NES (see Sects. 3.1 and 3.3),

The S-shape of the SIM suggests also the possible existence of *relaxation oscillations* (Grasman 1987), starting at $S_0 \in L_M$, the system jumps to S_1 , which is followed by a slow evolution of the trajectory of the system (in the stable domain of the SIM) until it reaches L_m . After another jump and a slow evolution (in the stable domain of the SIM), the trajectory returns to $S_0 \in L_M$. Such scenario of relaxation oscillations for the slow-flow system can explain the existence of *Strongly Modulated Responses* (Gendelman *et al.* 2010, Gendelman and Bar 2010, Starosvetsky and Gendelman 2008) for the HM+NES. Note that if $\mu > \omega_y/\sqrt{3}$, the S-shape nature of the SIM is lost and therefore relaxation oscillations are not possible.

Until a first jump the slow-flow evolves the same way as for relaxation oscillations mechanism. However, instead of moving toward a stable fixed point or the folded line L_m , the trajectory of the system follows the SIM to the infinity. This scenario explains *no suppression* regime for the HM+NES.

The existence of one of the three steady-state regimes described above or an other depends of the position and the stability of the fixed points of the full slow-flow system (47)-(49). Indeed, a

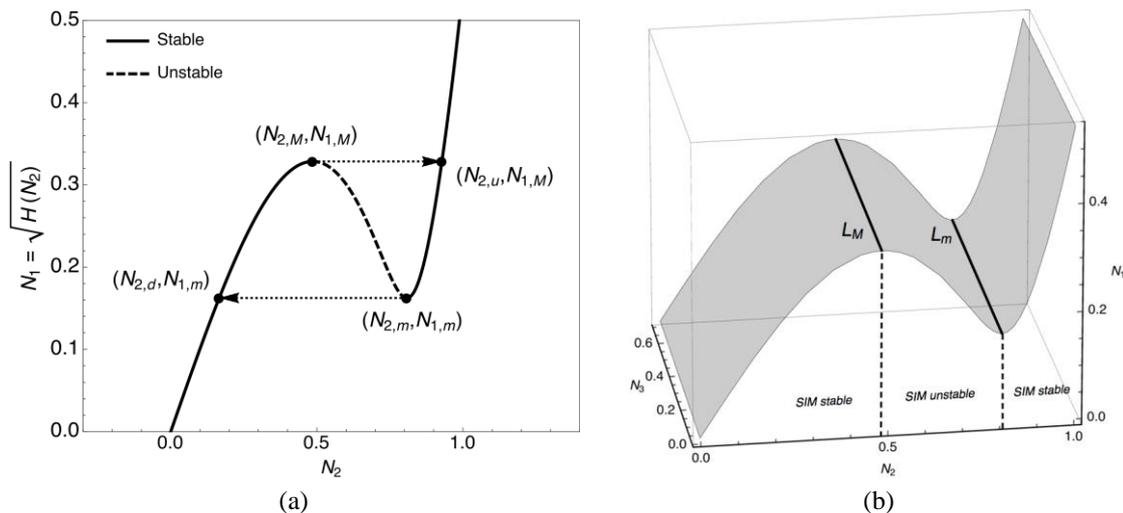


Fig. 4 Slow Invariant Manifold (SIM), Following parameters are used: $\omega_y=1$, $\alpha_3=2$ and $\mu=0.2$. (a) In the (N_1, N_2) -plan and (b) In the (N_1, N_2, N_3) -space

stable fixed point of the full slow flow placed on the stable part of the SIM is a necessary condition to obtain PRs of the HM+NES (20-22). On the other hand, the relaxation oscillations of the slow flow (or SRMs for the HM+NES) can exist if both folded lines L_M and L_m have attractive parts. Position and stability of the fixed points of the full slow-flow system and attractivity (or repulsively) of the folded lines are determined in the next section through the study of the super-slow time scale.

3.4.2 Super-slow time scale: ε^1 -order of the system

The expressions of order ε^1 correspond to the following differential equations

$$\frac{\partial \phi_1}{\partial t_1} = -\frac{\lambda_y}{2} \phi_1 - j \frac{\omega_y}{2} (\phi_1 - \phi_2) + S_d \omega_y \phi_3 \quad (75)$$

$$\frac{\partial \phi_2}{\partial t_1} = -\frac{\lambda_y}{2} \phi_1 - j \frac{\omega_y}{2} (\phi_1 - \phi_2) + S_d \omega_y \phi_3 - \frac{\mu}{2} \phi_2 + j \frac{3\alpha_3}{8\omega_y^3} \phi_2 |\phi_2|^2 \quad (76)$$

$$\frac{\partial \phi_3}{\partial t_1} = -\left(\frac{\lambda_\delta}{2} - ja\right) \phi_3 + \frac{S_c \omega_y^2}{8\omega_\delta} \phi_1. \quad (77)$$

We investigate the behavior of $\Phi_2(t_1)$ and $\Phi_3(t_1)$ with the super-slow time t_1 . The behavior of $\Phi_1(t_1)$ is related to that of $\Phi_2(t_1)$ through the SIM (65),

For this sake, we consider only Eqs. (75) and (77) in the limit $t_0 \rightarrow \infty$. Using Eqs (55) and (66) and the SIM Eq. (65), Eqs. (75) and (77) are written as follow

$$\frac{\partial [\Phi_2 F(|\Phi_2|)]}{\partial t_1} = -\frac{\lambda_y}{2} \Phi_2 F(|\Phi_2|) - j \frac{\omega_y}{2} \Phi_2 (F(|\Phi_2|) - 1) + S_d \omega_y \Phi_3 \quad (78)$$

$$\frac{\partial \Phi_3}{\partial t_1} = -\left(\frac{\lambda_\delta}{2} - ja\right) \Phi_3 + \frac{S_c \omega_y^2}{8\omega_\delta} \Phi_2 F(|\Phi_2|). \quad (79)$$

Introducing the polar coordinates

$$\Phi_2 = N_2(t_1) e^{i\Theta_2(t_1)} \quad \text{and} \quad \Phi_3 = N_3(t_1) e^{i\Theta_3(t_1)}, \quad (80)$$

and separating real and imaginary parts, Eqs. (78-79) takes the following form

$$F_R \frac{\partial N_2}{\partial t_1} - N_2 \frac{\partial \Theta_2}{\partial t_1} F_I + N_2 \frac{\partial N_2}{\partial t_1} F_{R'} = -\frac{\lambda}{2} N_2 F_R + \frac{\omega_y}{2} N_2 F_I + S_d \omega_y N_3 \cos(\Delta_{32}) \quad (81)$$

$$N_2 \frac{\partial \Theta_2}{\partial t_1} F_R + \frac{\partial N_2}{\partial t_1} F_I + N_2 \frac{\partial N_2}{\partial t_1} F_{I'} = -\frac{\lambda}{2} N_2 F_I - \frac{\omega_y}{2} N_2 (F_R - 1) + S_d \omega_y N_3 \sin(\Delta_{32}) \quad (82)$$

$$\frac{\partial N_3}{\partial t_1} = -\frac{\lambda}{2} N_3 + \frac{S_c \omega_y^2}{8\omega_\delta} N_2 (F_R \cos(\Delta_{32}) + F_I \sin(\Delta_{32})) \quad (83)$$

$$\frac{\partial \Theta_3}{\partial t_1} = a N_3 + \frac{S_c \omega_y^2}{8\omega_\delta} N_2 (F_I \cos(\Delta_{32}) - F_R \sin(\Delta_{32})), \quad (84)$$

involving the variable $\Delta_{32} = \Theta_3 - \Theta_2$. Combining Eqs. (81)-(84), the system can be finally reduced (after some calculation steps) to the following form

$$H'(N_2) \frac{\partial N_2}{\partial t_1} = f_{N_2}(N_2, N_3, \Delta_{32}) \quad (85)$$

$$H'(N_2) \frac{\partial \Delta_{32}}{\partial t_1} = f_{\Delta_{32}}(N_2, N_3, \Delta_{32}) \quad (86)$$

$$\frac{\partial N_3}{\partial t_1} = f_{N_3}(N_2, N_3, \Delta_{32}), \quad (87)$$

where

$$f_{N_2}(N_2, N_3, \Delta_{32}) = a_{11} \cos \Delta_{32} + a_{12} \sin \Delta_{32} - c_1, \quad (88)$$

$$f_{\Delta_{32}}(N_2, N_3, \Delta_{32}) = a_{21} \cos \Delta_{32} + a_{22} \sin \Delta_{32} - c_2, \quad (89)$$

$$f_{N_3}(N_2, N_3, \Delta_{32}) = a_{31} \cos \Delta_{32} + a_{32} \sin \Delta_{32} - c_3. \quad (90)$$

The coefficients a_{11} , a_{12} , a_{21} , a_{22} , a_{31} , a_{32} , c_1 , c_2 and c_3 are not specified here. Eqs. (85)-(87) admit 0, 1 or 2 fixed points denoted $\{N_2^e, N_3^e, \Delta_{32}^e\}$ and defined by

$$f_{N_2}(N_2^e, N_3^e, \Delta_{32}^e) = 0, \quad (91)$$

$$f_{\Delta_{32}}(N_2^e, N_3^e, \Delta_{32}^e) = 0, \quad (92)$$

$$f_{N_3}(N_2^e, N_3^e, \Delta_{32}^e) = 0 \quad (93)$$

$$H'(N_2^e) \neq 0. \quad (94)$$

If $\varepsilon=1$, fixed points computed from Eqs. (91)-(94) corresponds to fixed points of the system (40) obtained in Sect. 3.3. As usual, stability of the fixed points are found by looking the sign of the eigenvalues real parts of the Jacobian matrix of the vector function $F_1 = (f_{N_2}, f_{\Delta_{32}}, f_{N_3})$ evaluated at $\{N_2^e, N_3^e, \Delta_{32}^e\}$.

A more detailed analysis of Eqs. (85)-(87) is proposed in (Bergeot et al. 2016). In particular, folded singularities, which are signs of SMR in the system are highlighted and computed.

3.5 Prediction of the steady-state response regimes of the HM+NES

The prediction of the steady-state response regimes of the HM+NES (20-22) resulting from initial conditions not too far from the trivial equilibrium position is obtained checking first the local stability property of the trivial equilibrium point of Eqs. (20)-(22) (see Sect. 2.2) and using the asymptotic analysis of the full slow-flow system (47)-(49) to characterize the response regimes when the trivial equilibrium point of Eqs. (20)-(22) is unstable. From the asymptotic analysis of slow-flow system (47)-(49) performed in Sect. 3.4, we characterize

- the SIM (65) and its the fold points $N_{2,M}$ and $N_{2,m}$ (see Eqs. (70)) and the points $N_{2,d}$ and $N_{2,u}$ defined in Eqs. (73) and (74) respectively,

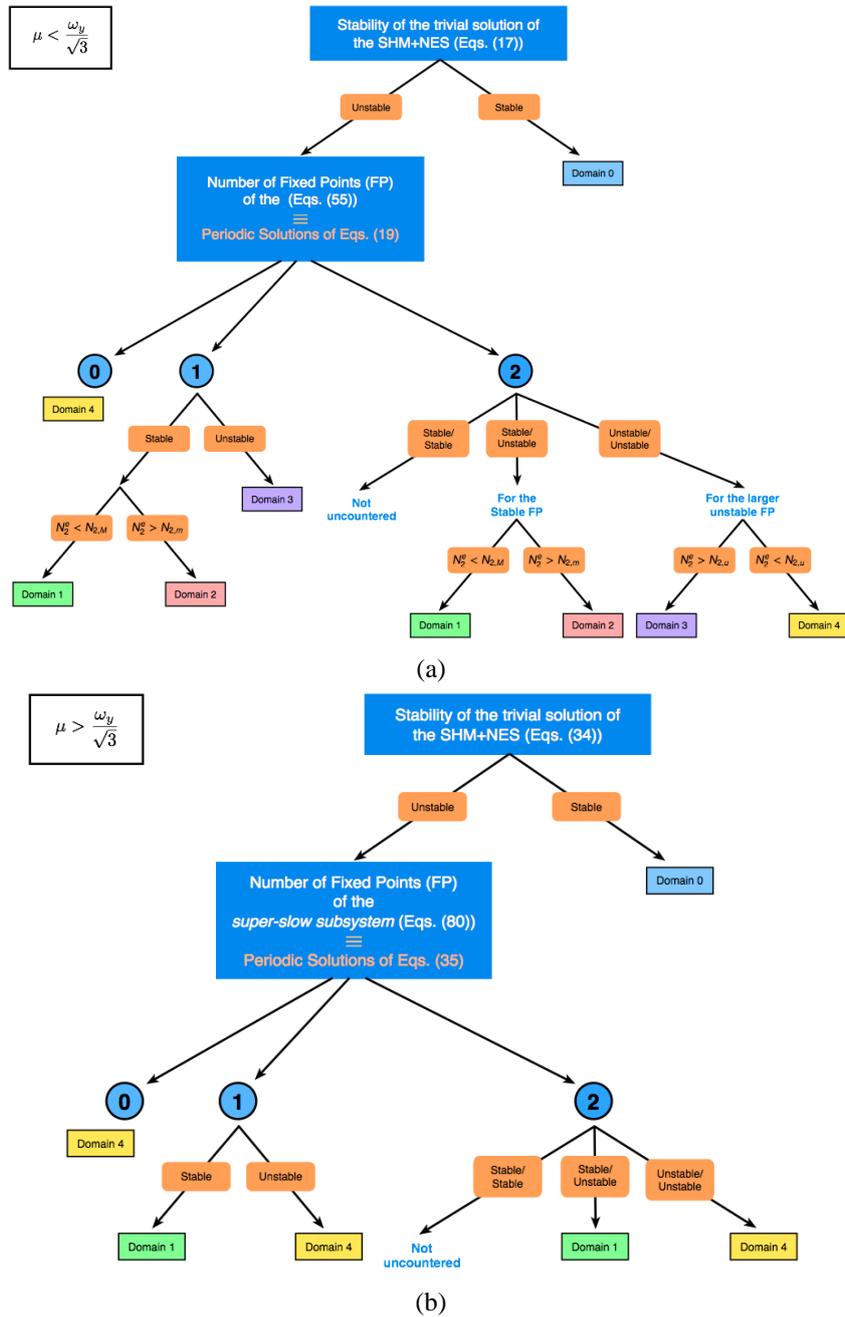


Fig. 5 Algorithm for the determination of the domain of existence of the steady-state regimes of the HM+NES (24-26), (a) $\mu < \omega_y/\sqrt{3}$ and (b) $\mu > \omega_y/\sqrt{3}$

- the stable periodic regimes (PRs) of Eqs. (20)-(22) as the stable fixed points of Eqs. (85)-(87),
- the non periodic response regimes (SMRs or no suppression) of Eqs. (20)-(22) from the unstable fixed points of Eqs. (85)-(87),

The steady-state response regimes are classified in five domains:

- Complete suppression (Domain 0)
- Partial suppression through PR (Domain 1)
- Partial suppression through PR or SMR (Domain 2)
- Partial suppression through SMR (Domain 3)
- No suppression (Domain 4)

The diagram of the Fig. 5 summarizes the algorithm used to, for a given set of parameters, the corresponding domain for the associated response regime: Fig. 5(a) for $\mu < \omega_y/\sqrt{3}$ and Fig. 5(b) $\mu > \omega_y/\sqrt{3}$. The algorithm is detailed in the following section.

4. Parametric study and numerical validation of the prediction algorithm

The procedure presented in Sect. 3.5 is used to analyze the influence of the damping μ of the NES and the rotor speed Ω , through the detuning parameter a , on the response regimes of the HM+NES model. The following set of numerical values of the parameters is used

$$\omega_y = 1, \omega_\delta = 2, \lambda_y = 0.3, \lambda_\delta = 0.15, S_d = 1, S_c = 2, \alpha_3 = 2, \varepsilon = 0.01, \tag{95}$$

with $\mu \in [0,1]$ and $a \in [-1.2,1.2]$. Results are plotted in Fig. 7 in which each domain is represented by an area of the plane (μ, a) . Finally, for each domain (except for Domain 0 which characterizes the *Complete suppression*) two examples are selected and studied deeply (see Figs. 8 to 13). The values of a and μ used for these examples and the corresponded coordinates N_1^e , N_2^e and N_3^e of the fixed points of (85-87) are indicated in Table 1. Table 2 shows the corresponded values of $N_{2,M}$, $N_{2,m}$, $N_{2,u}$ and $N_{2,d}$.

4.1 Domain 0: Complete suppression

Analytical prediction of the *complete suppression* is performed in Sect. 2.2. To obtain the Domain 0, the values of a that annul one of the eigenvalues Eqs. (20)-(22) are computed with

Table 1 Values of a and μ used in Examples 1a, 1b, 2a, 2b, 3a, 3b, 4a and 4b. Coordinates N_1^e , N_2^e and N_3^e of the corresponding fixed points of (85)-(87) are also indicated. S=stable and U=unstable

Example	a	μ	Number of Fixed Pt.	Fixed Pt. 1: $\{N_1^e, N_2^e, N_3^e\}$	Fixed Pt. 2: $\{N_1^e, N_2^e, N_3^e\}$
1a	-0.65	0.15	2	{0.312,0.406,0.211} S	{0.448,0.978,0.412} U
1b	0.3	0.7	1	{0.608,0.859,0.723} S	—
2a	0.2	0.45	2	{0.372,0.826,0.539} S	{151,4.7,75.4} U
2b	0.2	0.37	1	{0.298,0.799,0.468} S	{151,4.7,75.6} U
3a	0.4	0.2	1	{0.195,0.744,0.319} U	—
3b	-0.2	0.2	2	{0.239,0.692,0.307} U	{1.22,1.16,0.77} U
4a	0.65	0.3	2	{0.287,0.870,0.458} U	{0.331,0.593,0.340} U
4b	0.65	0.5	0	—	—

Table 2 Values of $N_{2,M}$, $N_{2,m}$, $N_{2,u}$ and $N_{2,d}$ corresponding to Examples 1a, 1b, 2a, 2b, 3a, 3b, 4a and 4b

Example	a	μ	$N_{2,M}$	$N_{2,m}$	$N_{2,u}$	$N_{2,d}$
1a	-0.65	0.15	0.479	0.812	0.935	0.124
1b	0.3	0.7		$\mu > \omega_y/\sqrt{3} \approx 0.577$		
2a	0.2	0.45	0.552	0.764	0.850	0.407
2b	0.2	0.37	0.523	0.784	0.886	0.321
3a	0.4	0.2	0.486	0.808	0.928	0.166
3b	-0.2	0.2	0.486	0.808	0.928	0.166
4a	0.65	0.3	0.505	0.796	0.908	0.254
4b	0.65	0.5	0.577	0.745	0.816	0.471

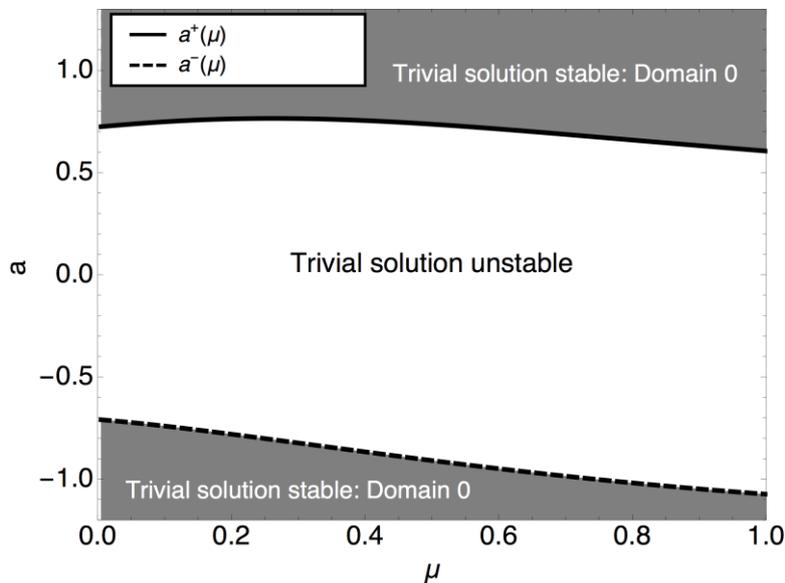


Fig. 6 Functions $a^+(\mu)$ and $a^-(\mu)$ and Domain 0 which corresponds to the region of stability of the trivial solution of Eqs. (20)-(22), Parameters used: see Eq. (95)

respect to the parameter μ . For each value of μ , two values of a annul the eigenvalue, defining the functions $a^+(\mu)$ and $a^-(\mu)$ in the plan (μ, a) (see Fig. 6), The area outside the curves $a^+(\mu)$ and $a^-(\mu)$ corresponds to stable trivial solution and in the area inside the curves, the trivial solution in unstable. The values $a^+(0)$ and $a^-(0)$ reflect the case without NES.

Remark. Following definitions correspond to situations for which $\mu < \omega_y/\sqrt{3}$. Otherwise, if $\mu > \omega_y/\sqrt{3}$, relaxation oscillations cannot exist. Therefore, only *Domains 1* and *Domains 4* are defined: *Domains 1* if one of the fixed point of slow-flow system is stable, *Domains 4* if not (see Fig. 5(b)),

4.2 Local stability of one of the fixed point of the slow-flow system: Domain 1 and 2

Fixed points of the slow-flow system correspond to periodic solutions of the HM+NES.

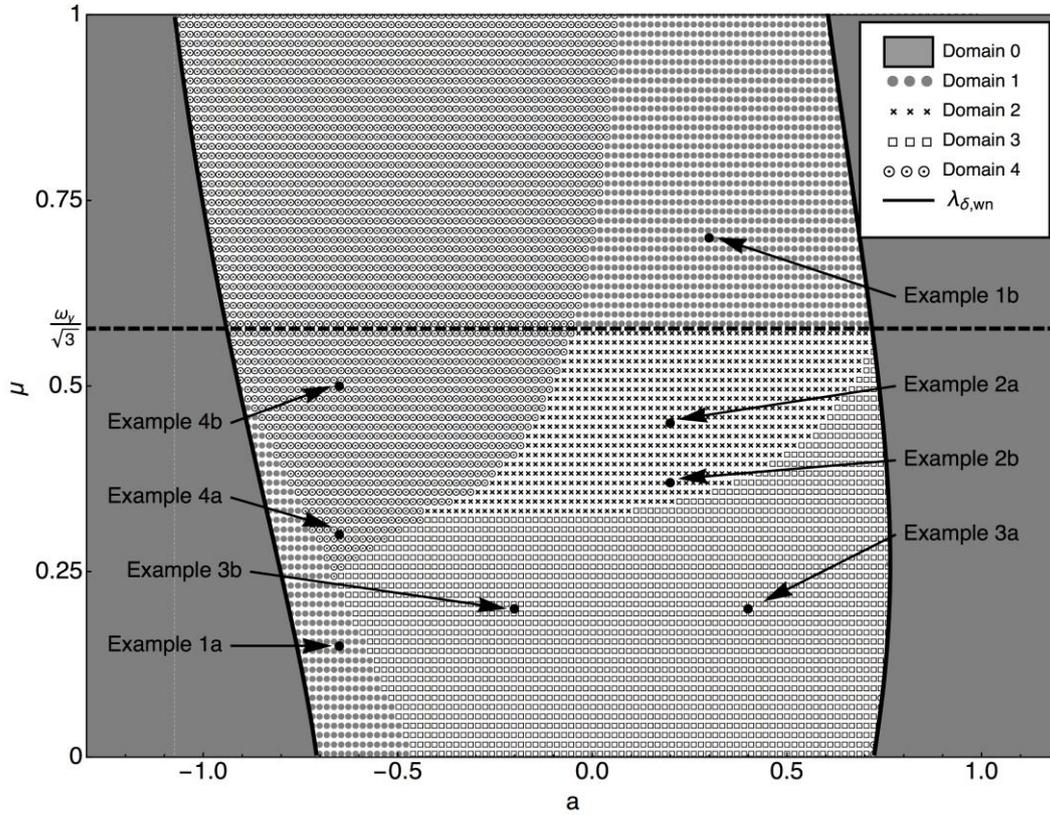


Fig. 7 Prediction, in the plane (a, μ) , of the domains of existence of the four steady-regimes. Domain 0: Complete suppression (the gray area), Domain 1: Partial suppression through PR (gray•), Domain 2: Partial suppression through PR or SMR (\times), Domain 3: Partial suppression through SMR (empty square), Domain 4: no suppression (empty circle with point), Parameters used: see Eq. (95)

Therefore, the domain of existence the *Partial suppression through Periodic Response* may correspond to the domain of local stability of one of the fixed point. However, the two following situations must be considered: $N_2^e < N_{2,M}$ and $N_2^e > N_{2,m}$ where N_2^e is the N_2 -coordinate of a stable fixed point. The two situations correspond to domain 1 and 2 respectively.

4.2.1 Domain 1: partial suppression through PR

This domain is represented by gray dots in Fig. 7. For $N_2^e < N_{2,M}$, the stable fixed point is reached before the folded line L_M . In this situation, relaxation oscillations or explosion of the slow-flow system and therefore SMR or *no suppression* regimes for the HM+NES are avoided. Therefore, domain 1 corresponds to *Partial suppression through Periodic Response*. Figs. 8 and 9 show two examples of this situation with $\mu < \omega_y/\sqrt{3}$ and $\mu > \omega_y/\sqrt{3}$ respectively.

4.2.2 Domain 2: partial suppression through PR or SMR

This domain is represented by black crosses in Fig. 7. The case of one stable fixed point which satisfies the condition $N_2^e > N_{2,m}$ highlights the limit of the local stability study. Indeed, in this

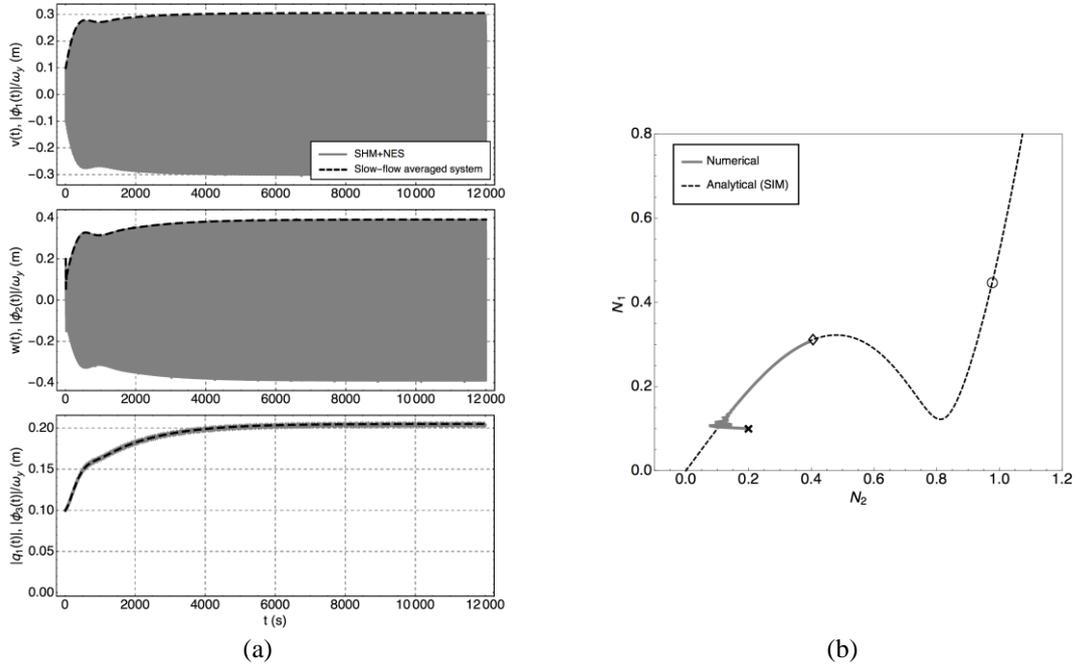


Fig. 8 Example 1a. Parameters used: see Eq. (95), $a=-0.65$ and $\mu=0.15$. (a) Comparison between numerical simulation of the HM+NES (24-26) (gray solid line) and the full slow-flow system (47)-(49) (black dashed line), (b) Comparison between the trajectory of the simulated slow-flow system (47)-(49) in the plane (N_2, N_1) and the *Slow Invariant Manifold* (65), “•”: position $(N_{2,M}, N_{1,M}), (N_{2,m}, N_{1,m}), (N_{2,u}, N_{1,u})$ and $(N_{2,d}, N_{1,d})$, \diamond : stable fixed points, “o”: unstable fixed points and \times : initial conditions

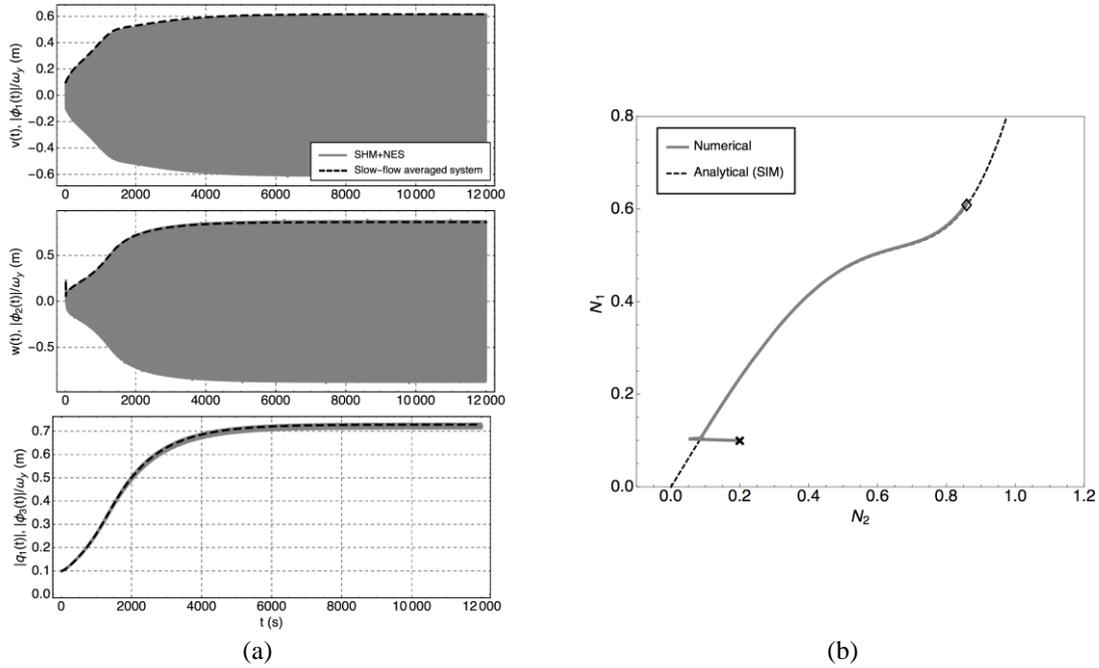


Fig. 9 Example 1b. Parameters used: see Eq. (95), $a=0.3$ and $\mu=0.7$. Same caption as for Fig. 8

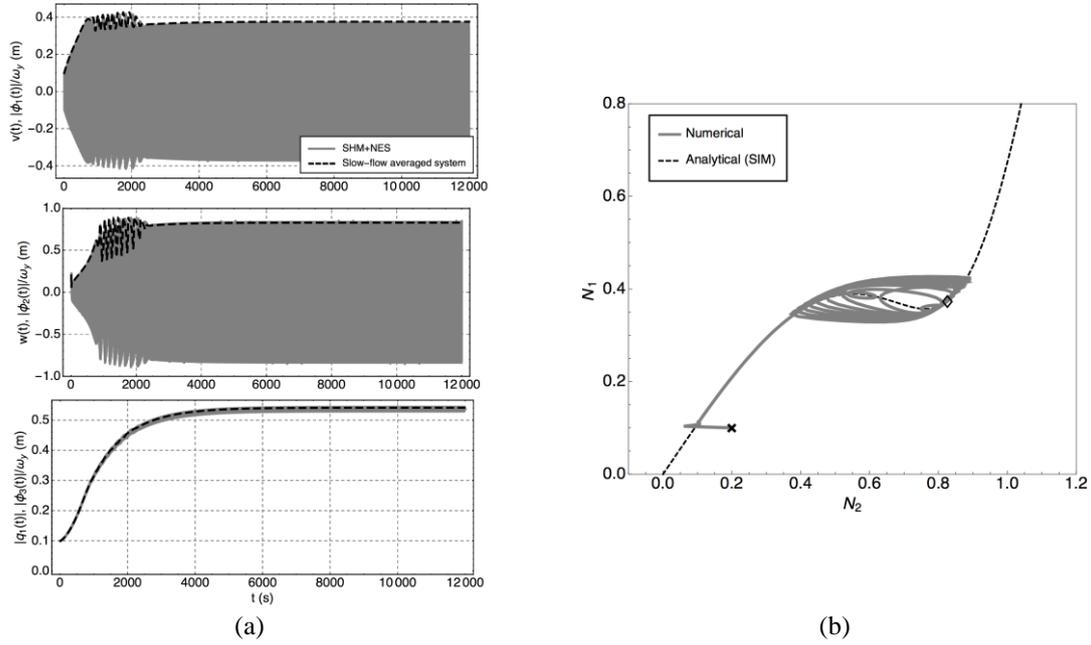


Fig. 10 Example 2a. Parameters used: see Eq. (95), $a=0.2$ and $\mu=0.45$. Same caption as for Fig. 8

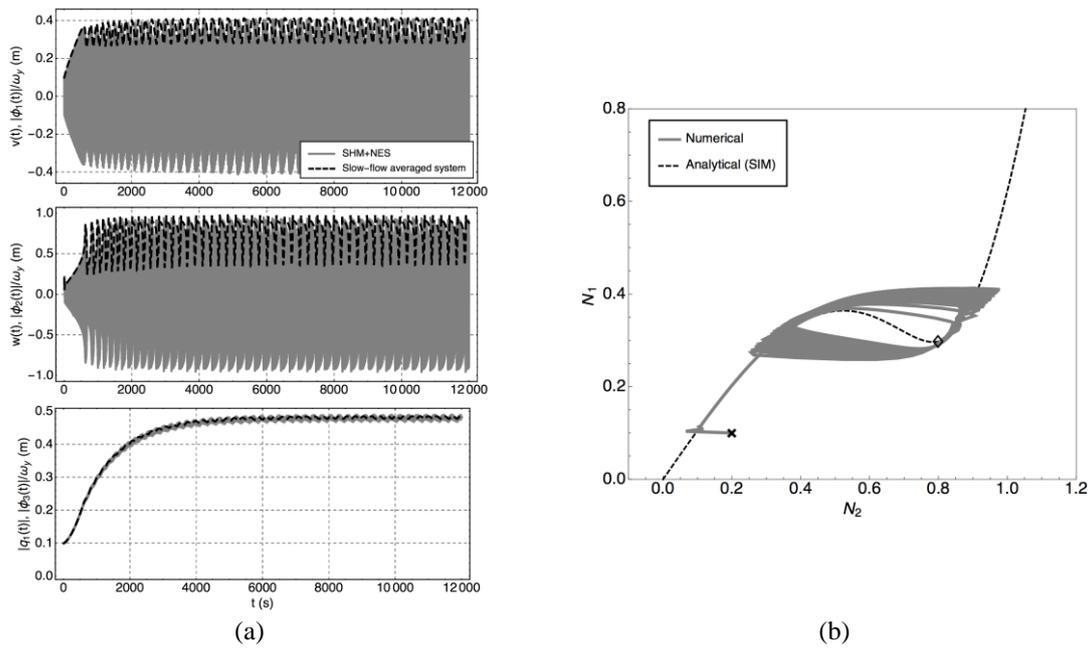


Fig. 11 Example 2b. Parameters used: see Eq. (95), $a=0.2$ and $\mu=0.37$. Same caption as for Fig. 8

case, at least one jump from $N_{2,M}$ to $N_{2,u}$ is needed to reach the fixed point. After that, the fixed point can be really reached or sustained relaxation oscillations of the the slow-flow system are observed. Examples of the two possible situations are shown in Figs. 10 (PR, Example 2a) and 11

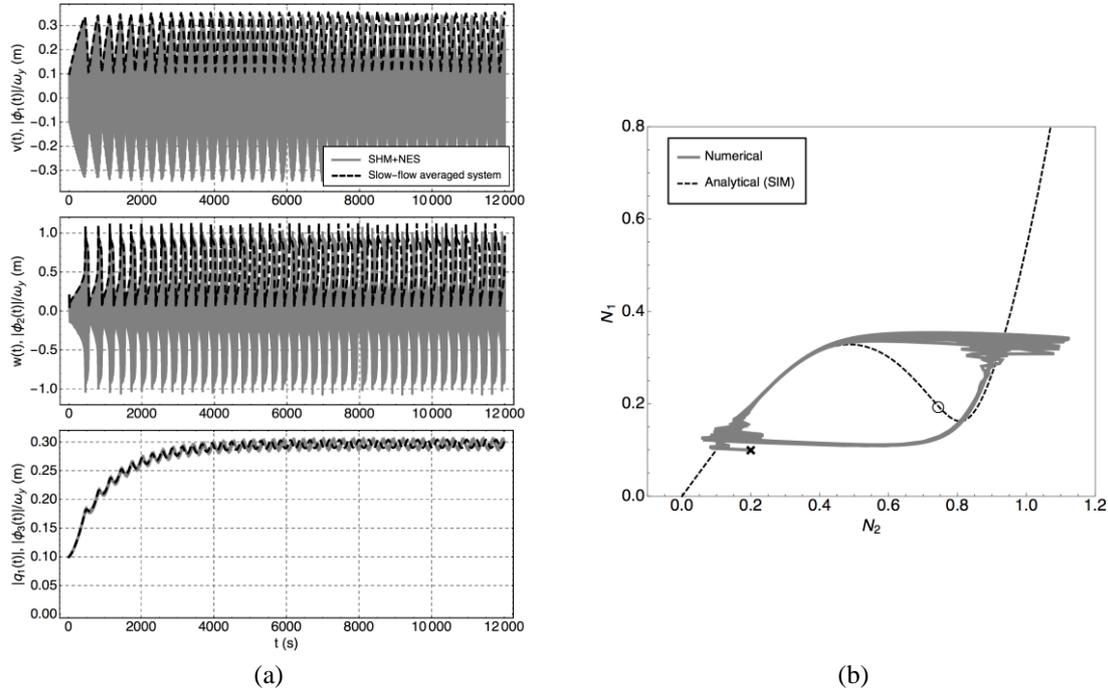


Fig. 12 Example 3a. Parameters used: see Eq. (95), $a=0.4$ and $\mu=0.2$. Same caption as for Fig. 8

(SMR, Example 2b),

4.3 Domain 3: partial suppression through SMR

This domain is represented by empty squares in Fig. 7 and corresponds to two situations. In the first situation, it exists one fixed point and it is unstable. In the second situation, there are two unstable fixed points and the larger of them should satisfied the following condition: $N_2^e > N_{2,\mu}$. Example 3a and Example 3b illustrate these two situations respectively (see Figs. 12 and 13), One can see in Figs. 12(b) and 13(b) that, in these situations, the fold points are reached by the system giving rise to relaxation oscillations of the slow-flow system which correspond to SMR for the HM+NES.

4.4 Domain 4: no suppression

This domain is represented by dotted circles in Fig. 7 and corresponds to two situations. The first situation corresponds to the case of two unstable fixed points and for both fixed points we have: $N_{2,M} < N_2^e < N_{2,\mu}$ (see Example 4a in Fig. 14), In the second situation, the slow-flow system has no fixed points. Therefore, there exists only the trivial solution of the HM+NES, and it is unstable (see Example 4b in Fig. 15), In the case of *no suppression*, the GR instability is too strong to be suppressed by the NES attachment through PRs or SMRs and after a transient regime an explosion of the slow-flow is finally observed.

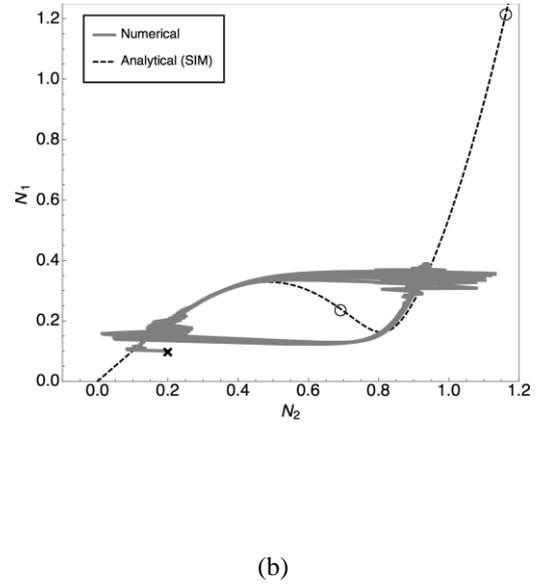
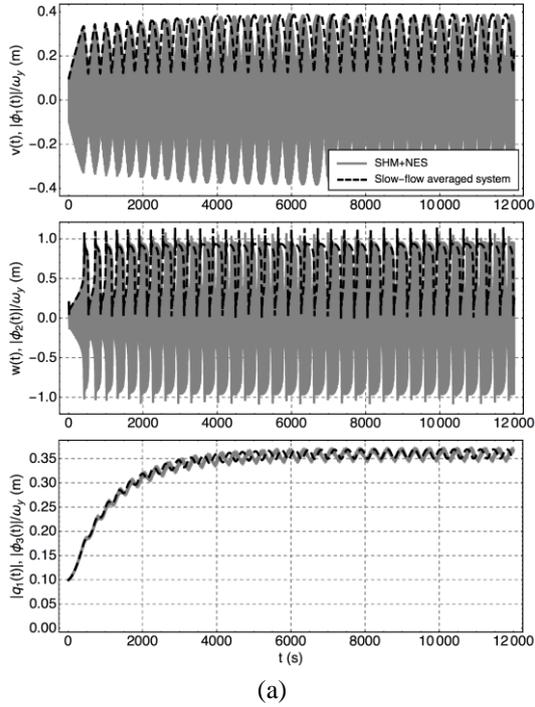


Fig. 13 Example 3b. Parameters used: see Eq. (95), $a=0.2$ and $\mu=-0.2$. Same caption as for Fig. 8

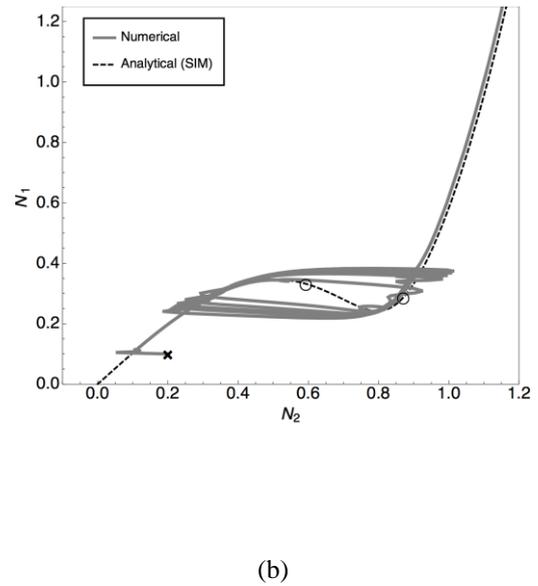
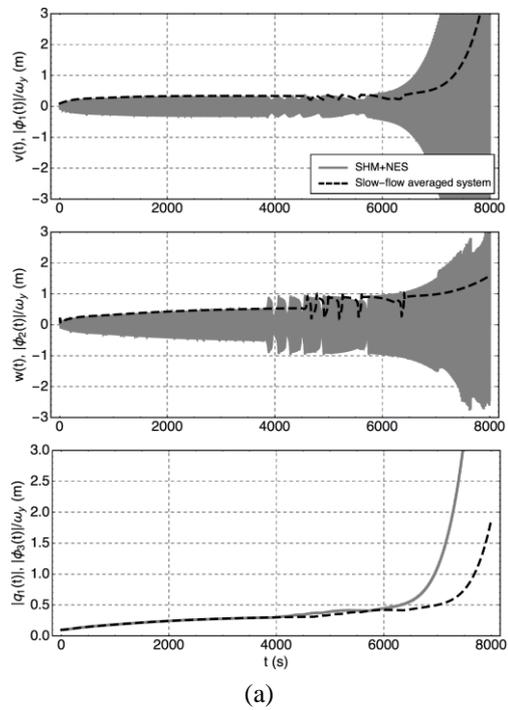


Fig. 14 Example 4a. Parameters used: see Eq. (95), $a=-0.65$ and $\mu=0.3$. Same caption as for Fig. 8

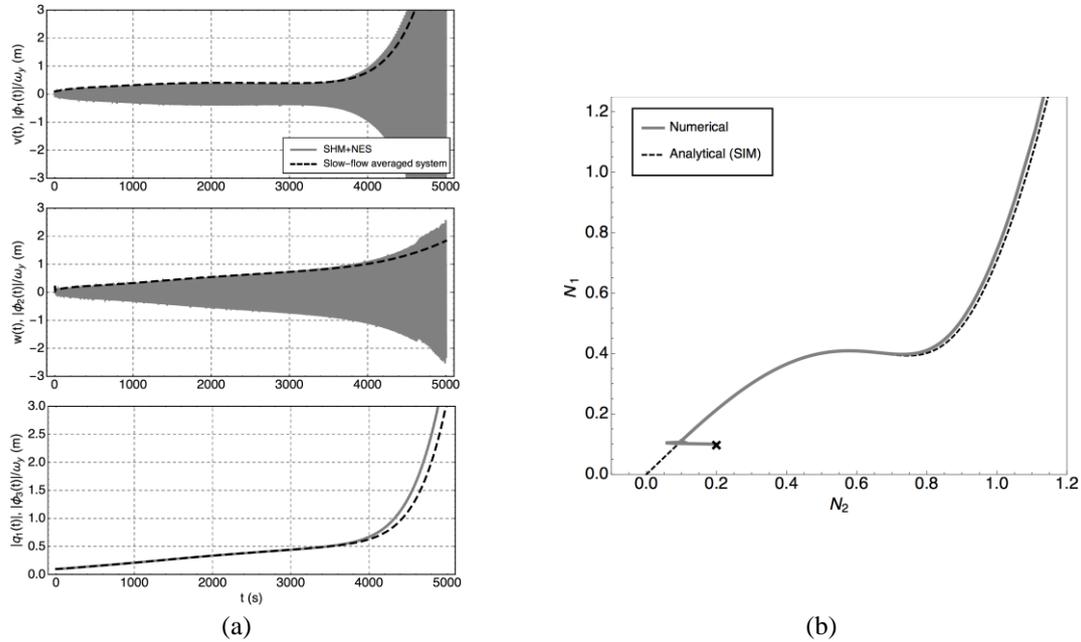


Fig. 15 Example 4b. Parameters used: see Eq. (95), $a=-0.65$ and $\mu=0.5$. Same caption as for Fig. 8

5. Conclusions

We studied the steady-state response regimes of a helicopter model reproducing GR instability coupled to ungrounded NES attached to the fuselage. An Helicopter Model (HM) was defined as a linear system involving blade and fuselage dynamics resulting from Coleman and binormal transformations. This model reproduces the GR instability corresponding to frequency coalescence of the fuselage mode and the regressive rotor mode. The HM is coupled to an ungrounded cubic NES defining the HM+NES (Helicopter Model+Non Linear Energy Sink) model. To analyze the steady-state response regimes, the system is partitioned in slow-fast dynamics using complexification-averaging approach. The presence a small dimensionless parameter related to the mass of the NES in the slow-flow system implies that it involves one $\hat{\epsilon}$ •slow $\hat{\epsilon}$ • complex variable and two $\hat{\epsilon}$ •super-slow $\hat{\epsilon}$ • complex variables. The $\hat{\epsilon}$ •super-slow/slow $\hat{\epsilon}$ • nature of the system allowed us to use multiple scale approach to analyze it. In particular, the Slow Invariant Manifold of the slow flow was determined. Its shape involving two folded lines and the associated stability properties provide an analytical tool to explain and predict the existence of three regimes: periodic response regimes, strongly modulated response regimes and no suppression regimes that appear when the trivial solution is unstable. A procedure that allows determining the domains of existence of these regimes was proposed. This procedure was used to analyze the influence of the damping of the NES and the rotor speed on the response regimes of the HM+NES model for a set of nominal numerical values of the other parameters of the model. In the unstable trivial solution area, four regimes were predicted: partial suppression through periodic response, partial suppression through strongly modulated response, partial suppression through periodic response or strongly modulated response and no suppression. All these regimes were validated from direct numerical integration of the HM+NES model.

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